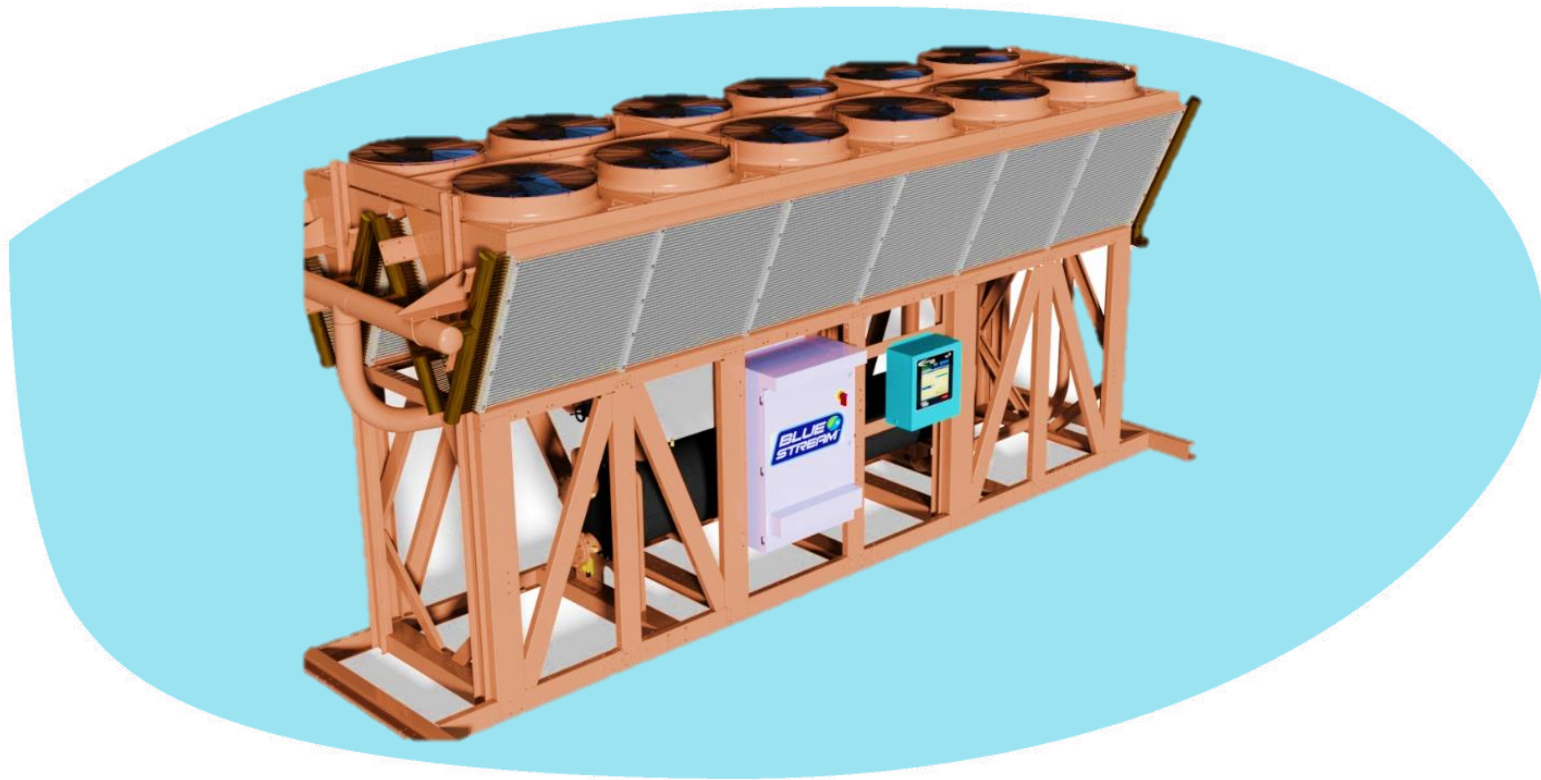


# Options for Reducing Water Consumption and Improving Operational Resiliency in Central Chiller Plants

*Water-Smart and Energy-Smart Heat Rejection*



Thomas P. Carter, P.E.  
Johnson Controls, Inc.  
[thomas.p.carter@jci.com](mailto:thomas.p.carter@jci.com)  
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**CAMPUS ENERGY 2016**

The Changing  
Landscape

February 8-12, 2016 | JW Marriott Austin Hotel | Austin, TX



# Acknowledgements:

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---

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- John Vucci
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- Lynne Harrahy
- Bryan Birosak

- University of Nebraska – Lincoln, NE



- Rhett Zeplin

- Michigan State University – East Lansing, MI



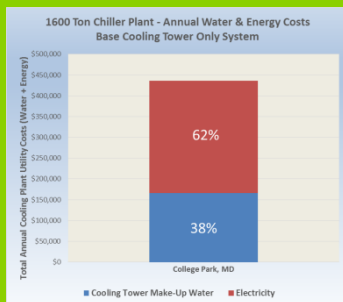
- Stacy Nurenberg

- Johnson Controls

- Zan Liu, Ph.D.

# Four Key Points to Remember

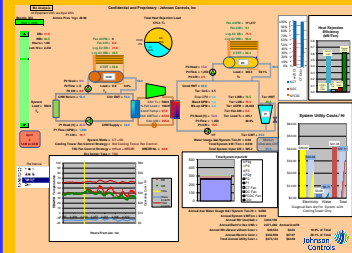
Water Costs  
Are Becoming  
An Increasing  
Larger  
Component of  
a Chiller  
Plant's Total  
Operating  
Cost



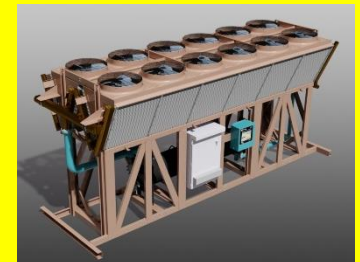
Drought and  
Water  
Availability  
Can Pose A  
Risk For  
Chiller Plant  
Operations



Analysis of  
Alternatives  
Requires a  
Thorough  
Annual  
System  
Evaluation



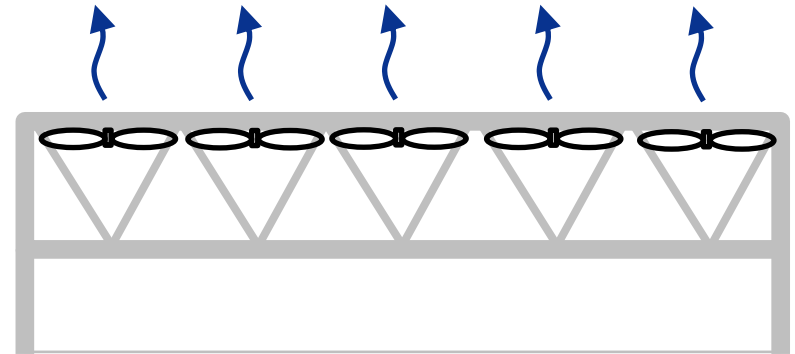
Hybrid  
Systems  
Offer a Cost  
Effective  
Way to  
Reduce  
Chiller Plant  
Water Use



# Air and Water Cooled Heat Rejection System Options

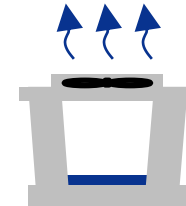
## Air-Cooled System

- Design day is based on **DRY BULB** temperature
- Consumes no water (no evaporative cooling)
- Large footprint / Requires very large airflow rates



## Water-Cooled System

- Design day is based on the lower **WET BULB** temperature
- Evaporative cooling process uses water to improve cooling efficiency
  - **80% LESS AIR FLOW** → Lower Fan Energy
  - Lower cost and smaller footprint
- Colder heat rejection temperatures improve system efficiency

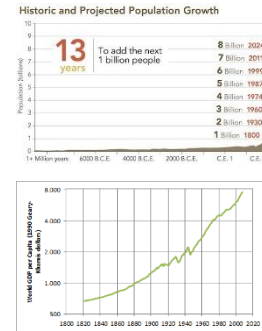


***However, water cooled systems depend on a reliable, continuous source of low cost water***

# Freshwater Stress - The Global Perspective

## Forces Driving Fresh Water Consumption:

- Population growth increases total demand
- Economic growth increases per capita demand

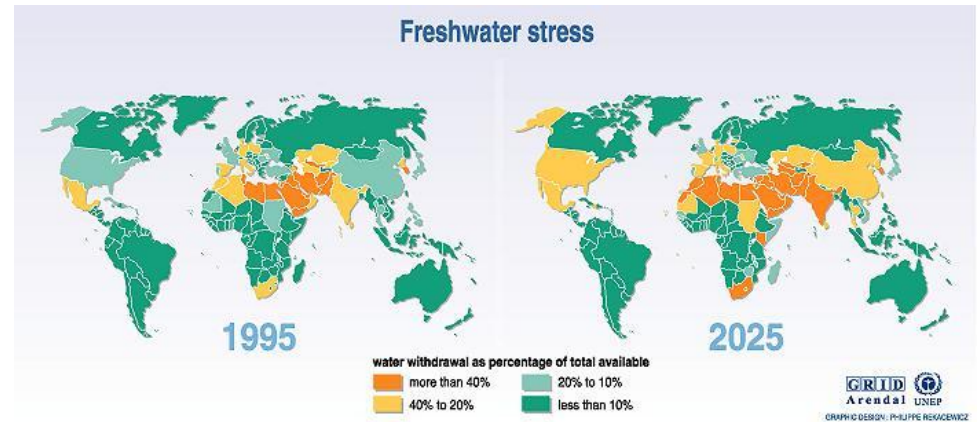
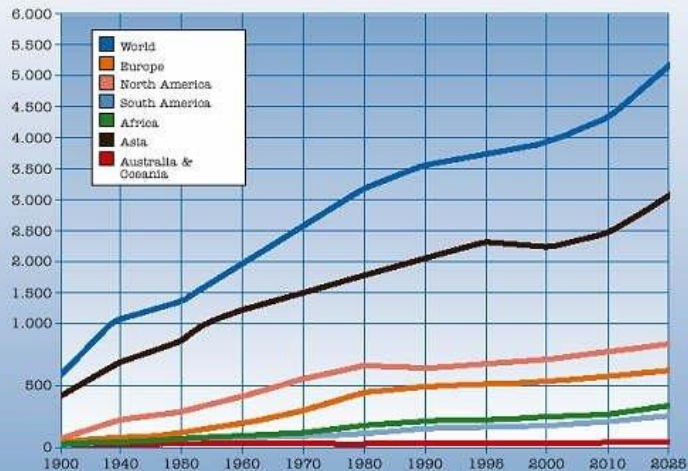


*Consumption increases ...*

*driving Freshwater Stress worldwide*

### Global Water Consumption 1900 - 2025

(by region, in billion m<sup>3</sup> per year)



*When the well's dry we know the worth of water.*  
*- Benjamin Franklin, 1746*



# Freshwater Stress – Increasing Prices and Concerns About Continuous Availability

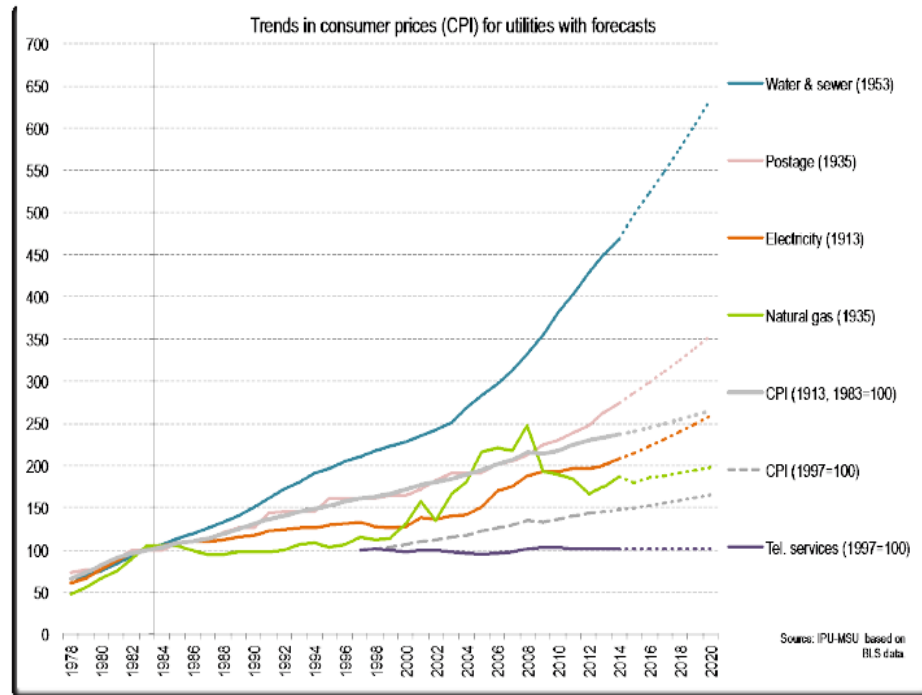
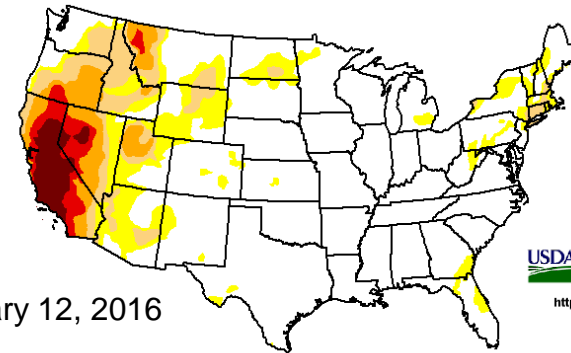


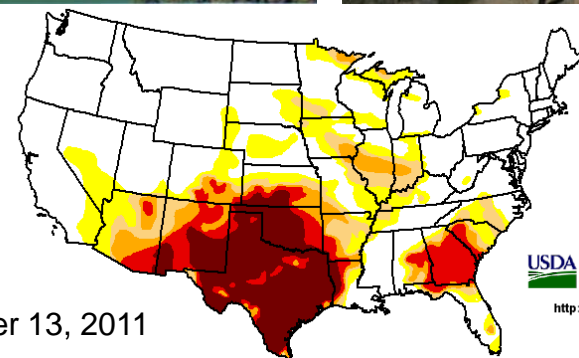
Exhibit 3. Trends in the Consumer Price Index for utilities (1978-2014, with forecasts). The index is set to 100 for 1982-1984 except for telephone services, where the index is set to 100 for 1997. Year (\*) indicates start of series. Heuristic forecasts are based on auto-regressive, integrated, moving average (ARIMA) methodologies.

© Beecher, Institute of Public Utilities, MSU [2015]

[4]

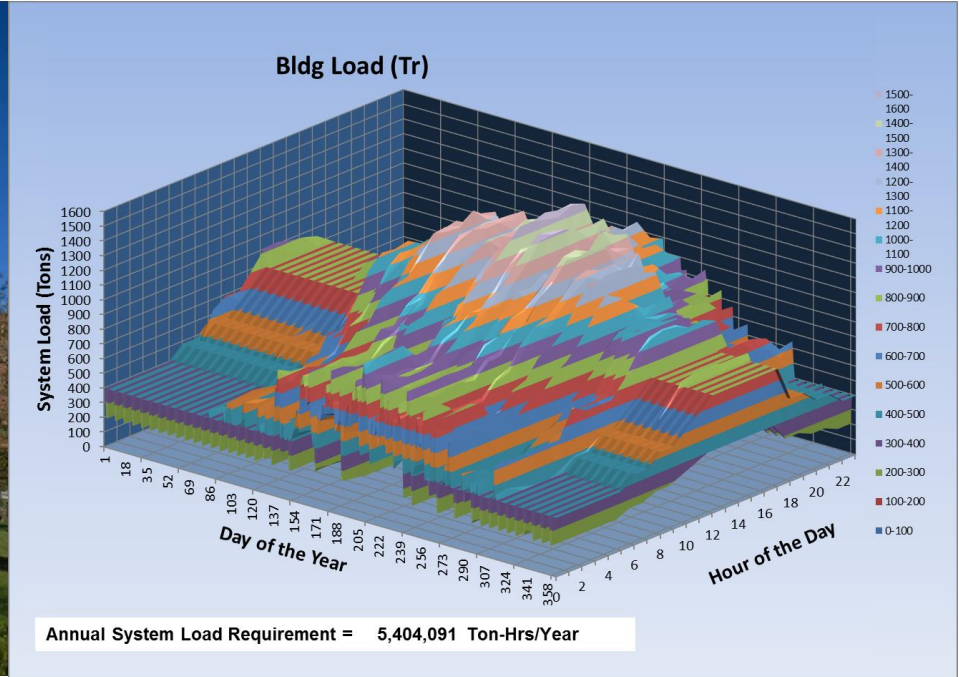


USDA  
National Drought Watch Center  
<http://droughtmonitor.unl.edu/>



USDA  
National Drought Watch Center  
<http://droughtmonitor.unl.edu/>

# University of Maryland College Park – Physical Sciences Building



# Model Assumptions

## Energy

Energy	\$0.0809	\$/kWh
Monthly Demand	\$5.28	\$/kW

## Water Related Costs

Make-up	\$ 7.29	\$/1000 gal
Sewer		
Blowdown	\$10.70	\$/1000 gal
Evaporation	\$10.06	\$/1000 gal
Chem. Treatment	\$ 2.78	\$/1000 gal Blowdown
CoC	4.5	
Fully Burdened	\$18.11	\$/1000 gal of Mk-Up

## Chillers

Type	Qty	kW / Ton
Water Cooled	2	0.579
Air Cooled	4	1.216

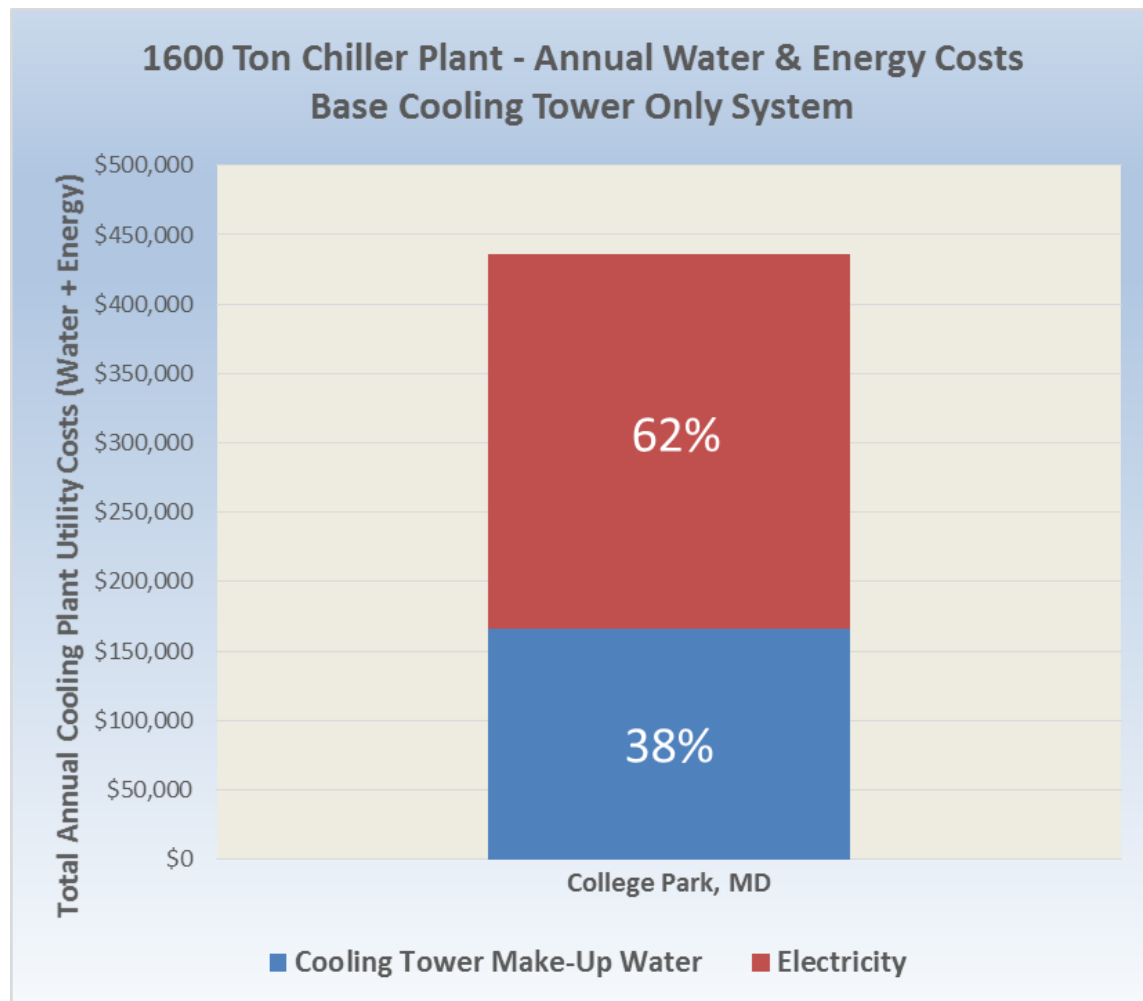
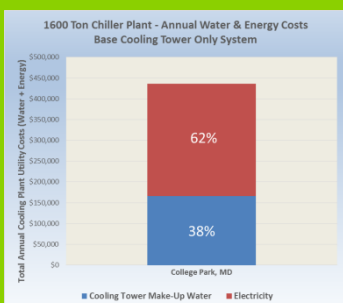
## Other Assumptions:

- 42°F Chilled Water Supply
- 2.0 GPM/Ton Chilled Water Flow Rate
- 3.0 GPM/Ton Condenser Water Flow Rate
- Cooling Tower Sized to Produce 85°F Condenser Water at the Summer Design WB



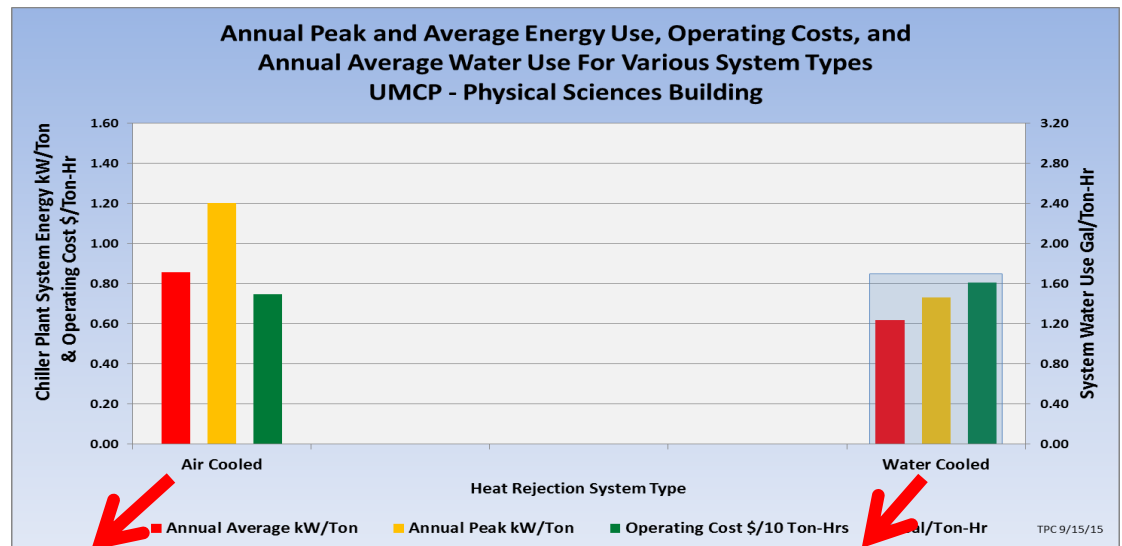
# Water & Waste Water Costs Represent A Growing Portion of Total Utility Spend for Many Chiller Plants

Water Costs  
Are Becoming  
An Increasing  
Larger  
Component of  
a Chiller  
Plant's Total  
Operating  
Cost



# Air-Cooled System vs Water-Cooled System

## UMCP Physical Sciences Building

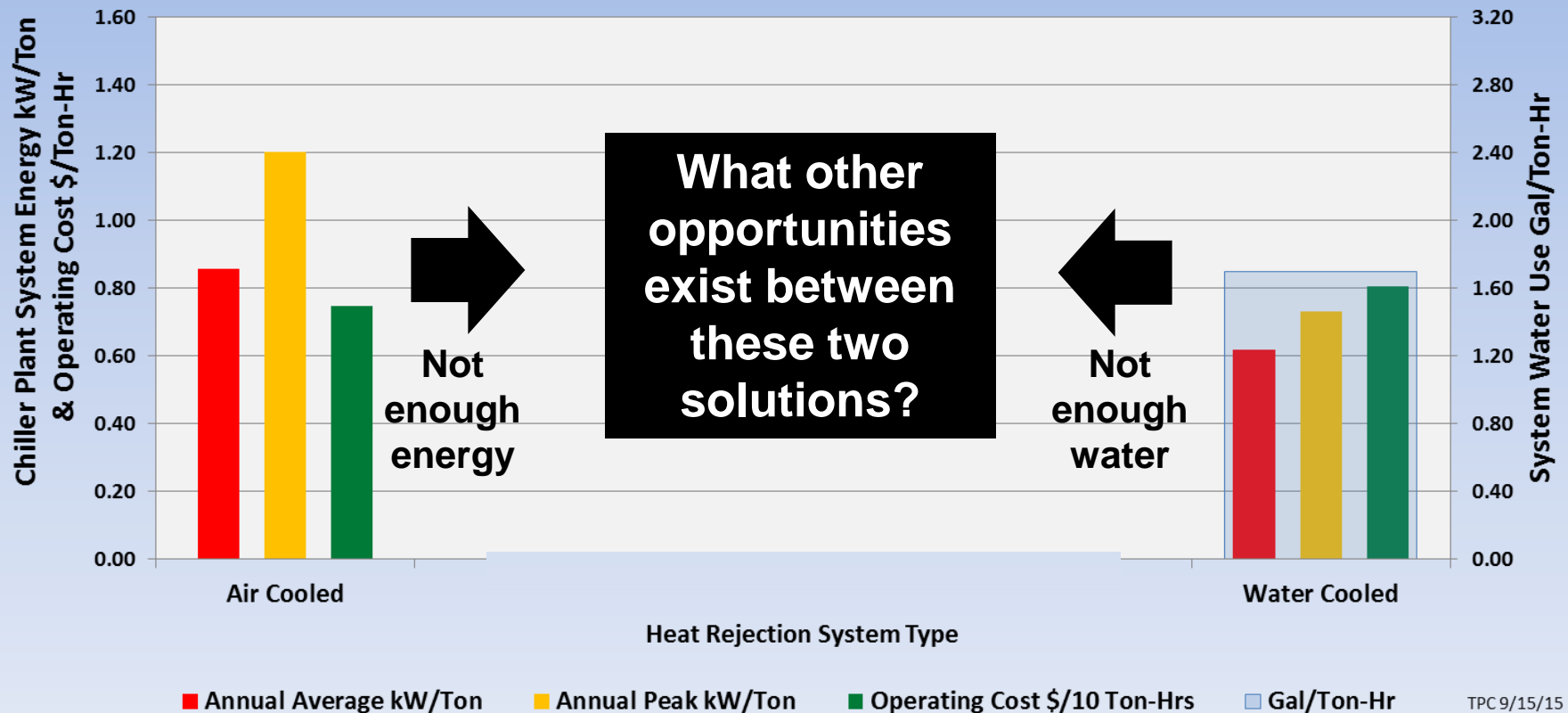


System Metrics	Air Cooled System	Compared to Water Cooled	Water Cooled System
Average kW / Ton	.857	+38.9%	.617
Peak kW / Design Ton	1.203	+65.0%	.729
Operating Cost \$ / 10 Ton-Hrs	\$.747	-7.3%	\$.806
Water Use Gal / Ton-Hr	0	-100%	1.697
Gal / Year	0	-9,171,760	9,171,760

# Air-Cooled System vs Water-Cooled System

## UMCP Physical Sciences Building

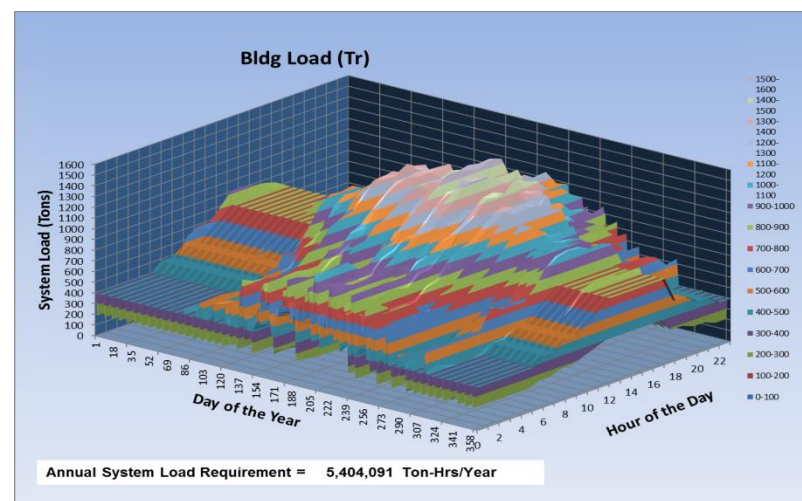
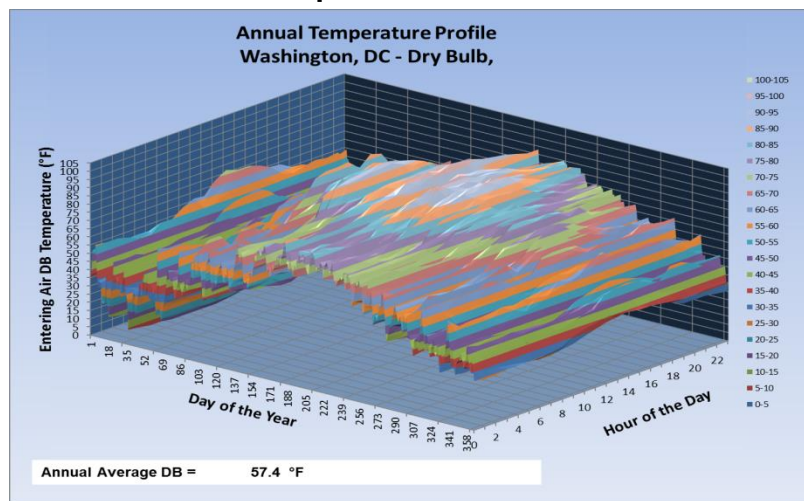
Annual Peak and Average Energy Use, Operating Costs, and Annual Average Water Use For Various System Types  
UMCP - Physical Sciences Building



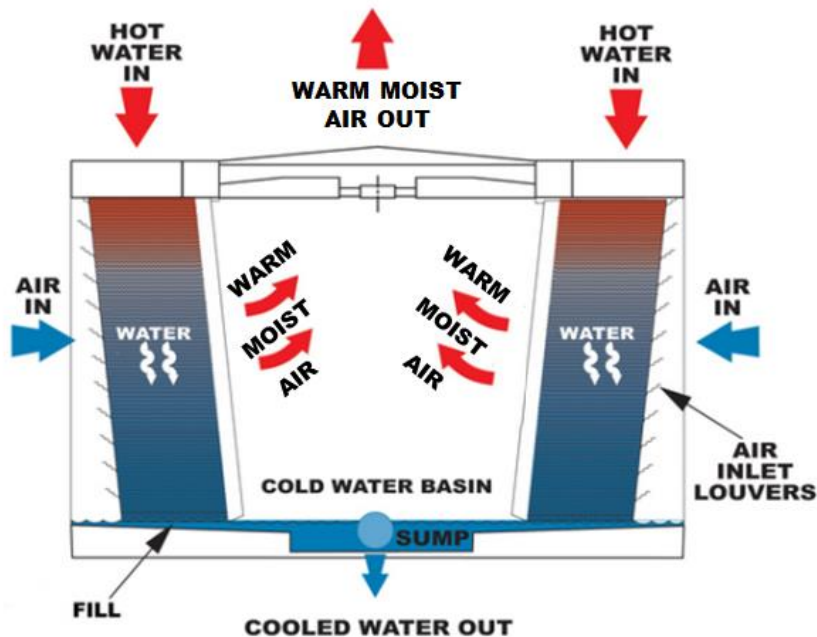
# Weather and Load Variations Provide Opportunities for Hybrid Wet / Dry Solutions

## Basic Principles:

- Operates **wet** during peak design periods to save energy (high temperatures and loads)
- Operates **dry** during low design periods to save water (lower temperatures and loads)
- Depending on the system design may either operate as **wet** or **dry** or may be able to operate both **wet** and **dry**



# The Open Cooling Tower is Very Efficient and It's Desirable to Have it as a Key Component of a Heat Rejection System



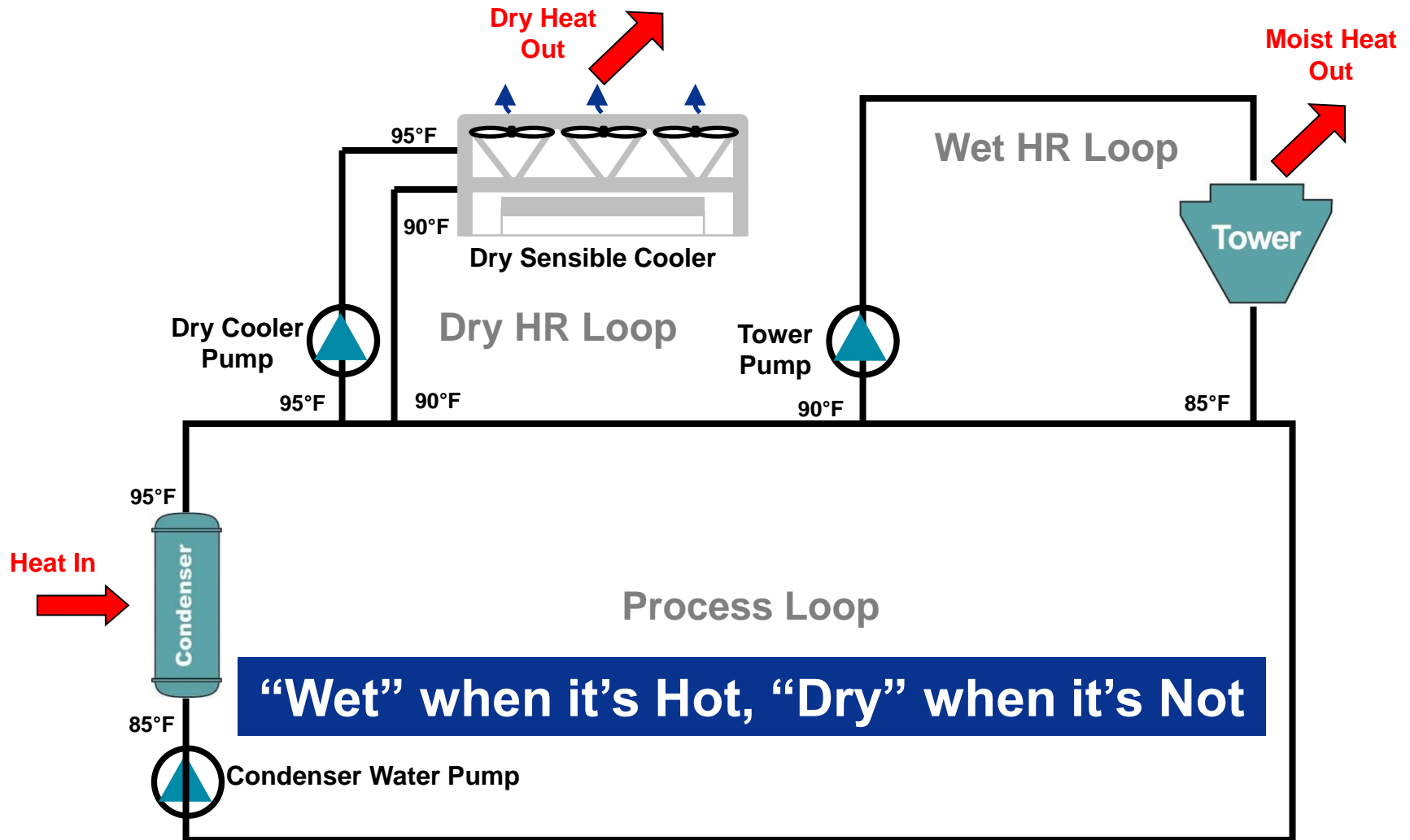
- Highly efficient – has the ability to saturate the exit air stream with moisture
  - Uses about 80% less air
  - Significantly lower cost
  - Significantly smaller footprint
  - Significantly lower fan energy
- Operates against the lower WB temperature sink

## ***The Challenge:***

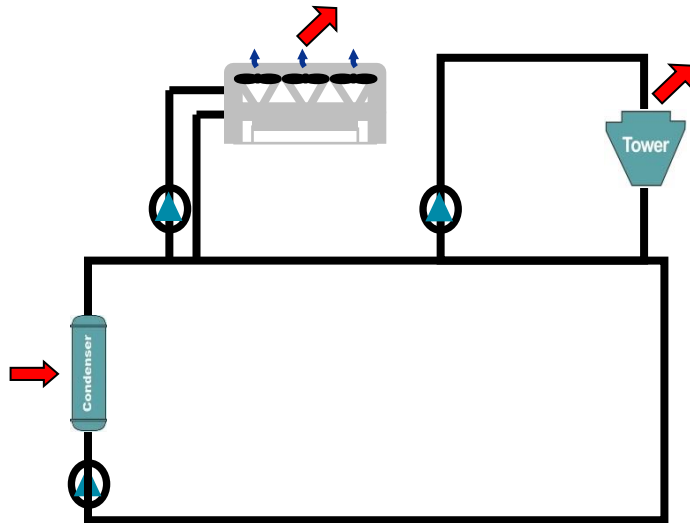
***How can the efficiency and capacity advantages of Evaporative Heat Rejection be delivered with far less water consumption?***



# Series Flow Dry / Wet Hybrid Heat Rejection System

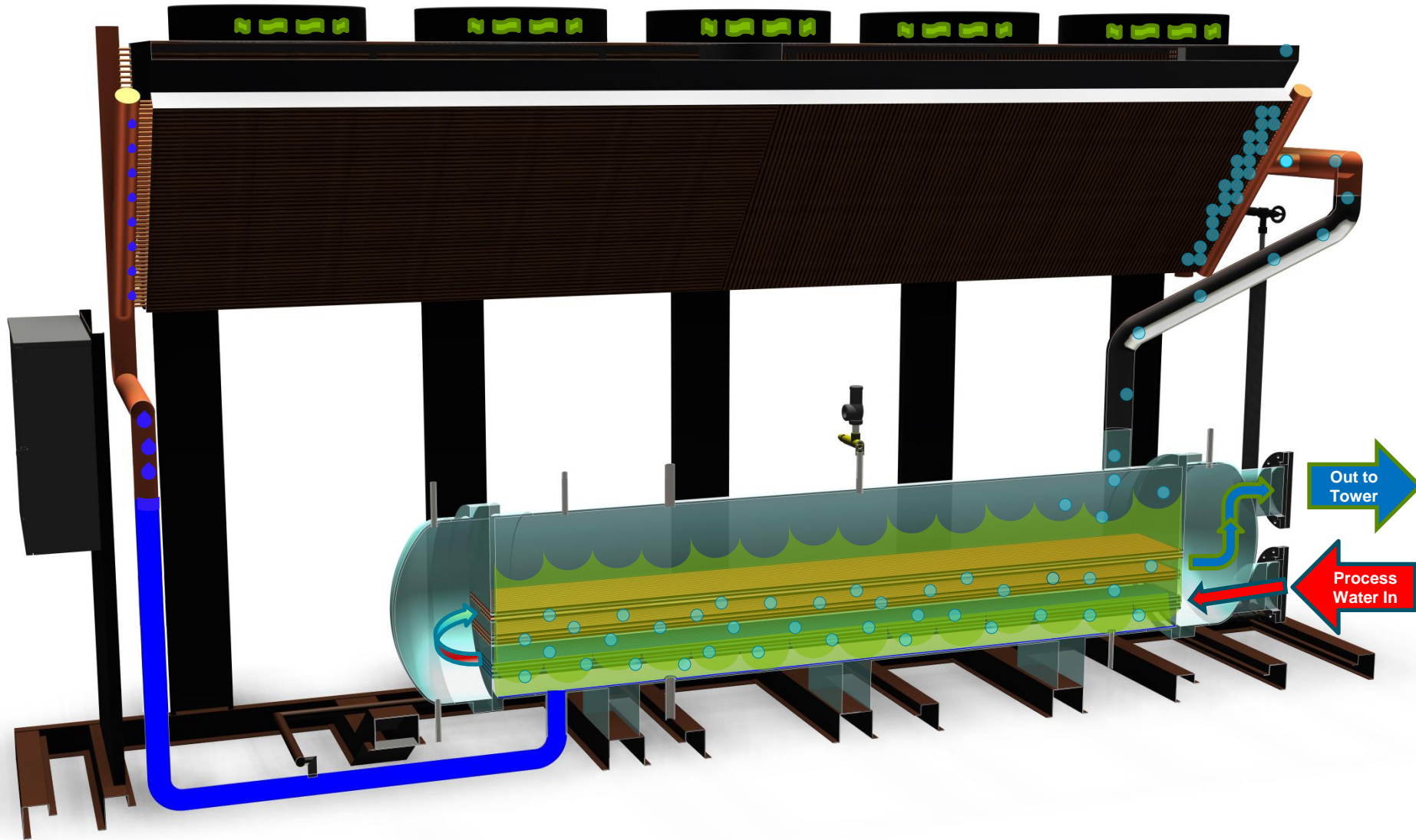


# Dry Sensible Heat Exchanger Requirements



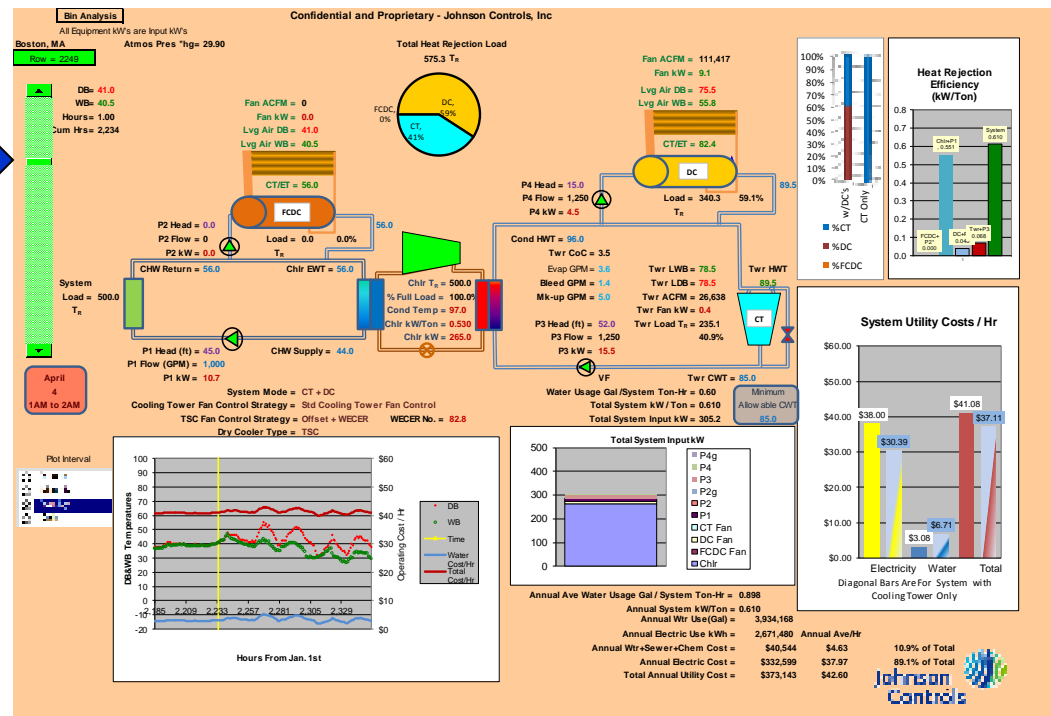
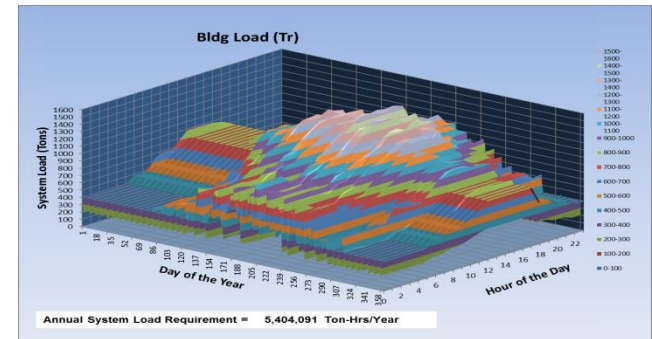
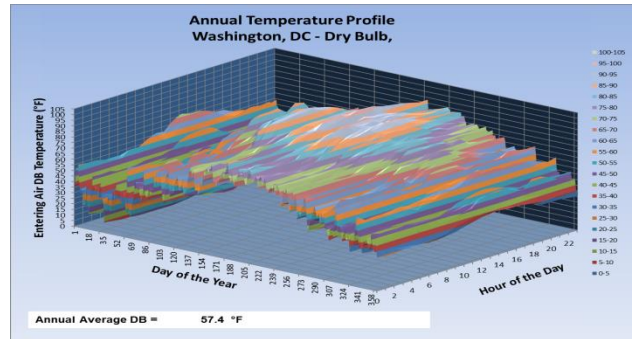
- Seems simple enough but ...
  - Open system – cleanability issues, material compatibility issues
  - Requires low pressure drop design
  - Control issues:
    - Percentage of cooling by each device
    - Optimum condenser entering water temperature
  - Freeze protection

# Thermosyphon Cooler – Conceptual Design



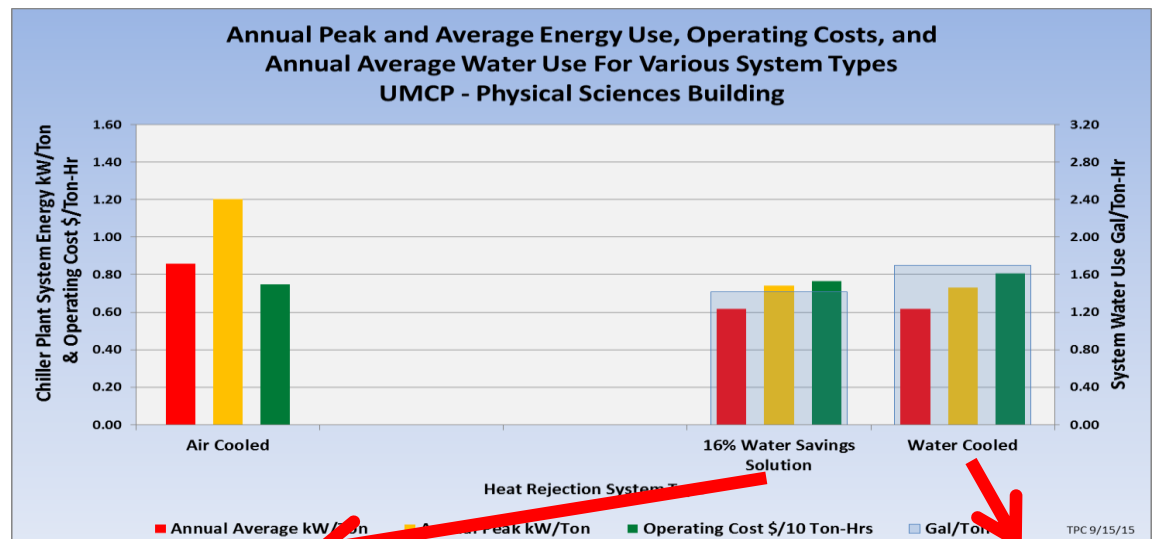
# Interactive System Schematic From The Chiller Plant Simulation Program

Analysis of Alternatives Requires a Thorough Annual System Evaluation



# 16% Water Savings TSC Hybrid System Example

- One TSC Unit
- WECER Control
- Minimum Condenser Water Temperature = 55°F

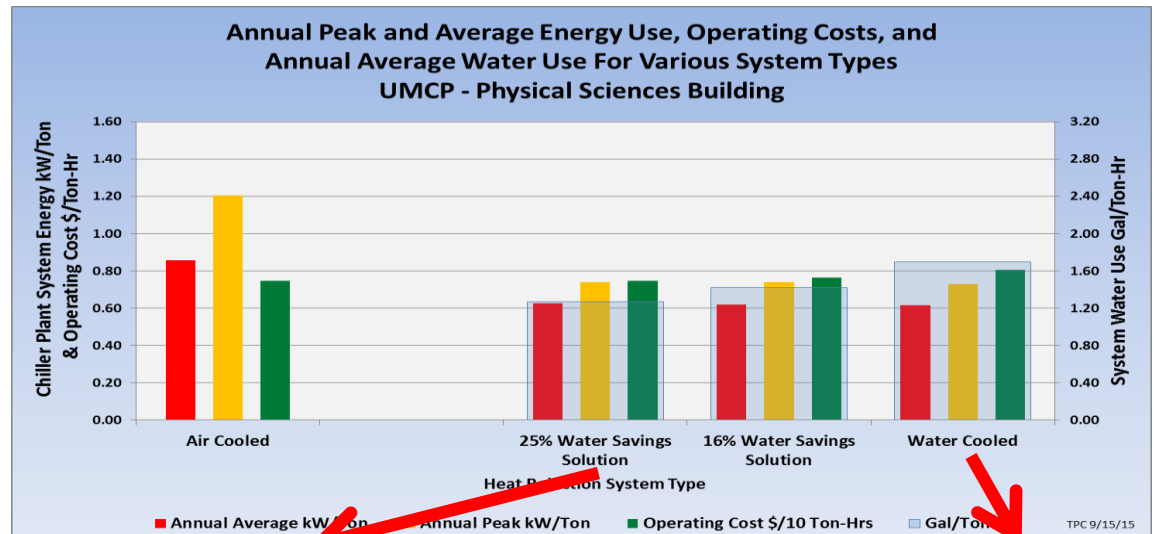


System Metrics	16% TSC Hybrid System	Compared to Water Cooled	Water Cooled System
Average kW / Ton	.618	+0.2%	.617
Peak kW / Design Ton	.740	+1.5%	.729
Operating Cost \$ / 10 Ton-Hrs	\$ .765	-5.2%	\$ .806
Water Use Gal / Ton-Hr Gal / Year	1.420 7,675,826	-16.3% <b>-1,495,934</b>	1.697 9,171,760



# 25% Water Savings TSC Hybrid System Example

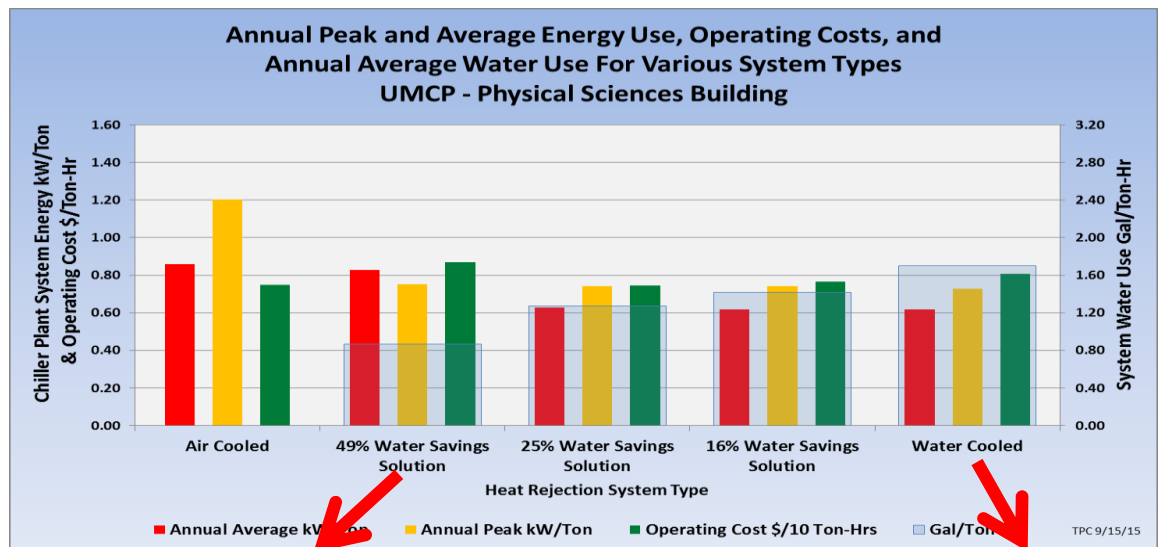
- **Two TSC Unit's**
- **WECER Control**
- **Minimum Condenser Water Temperature = 55°F**



System Metrics	25% TSC Hybrid System	Compared to Water Cooled	Water Cooled System
Average kW / Ton	.628	+1.8%	.617
Peak kW / Design Ton	.740	+1.5%	.729
Operating Cost \$ / 10 Ton-Hrs	\$.746	-7.4%	\$.806
Water Use Gal / Ton-Hr Gal / Year	1.271 6,869,141	-25.1% <b>-2,301,819</b>	1.697 9,171,760

# 49% Water Savings TSC Hybrid System Example

- Two TSC Unit's
- **Max Water Savings Control Mode**
- Minimum Condenser Water Temperature = **85°F**



System Metrics	49% TSC Hybrid System	Compared to Water Cooled	Water Cooled System
Average kW / Ton	.827	+34.0%	.617
Peak kW / Design Ton	.751	+3.0%	.729
Operating Cost \$ / 10 Ton-Hrs	\$.867	+7.6%	\$.806
Water Use Gal / Ton-Hr Gal / Year	0.867 4,686,357	-48.9% <b>-4,485,403</b>	1.697 9,171,760

# Comparisons Among Several Universities

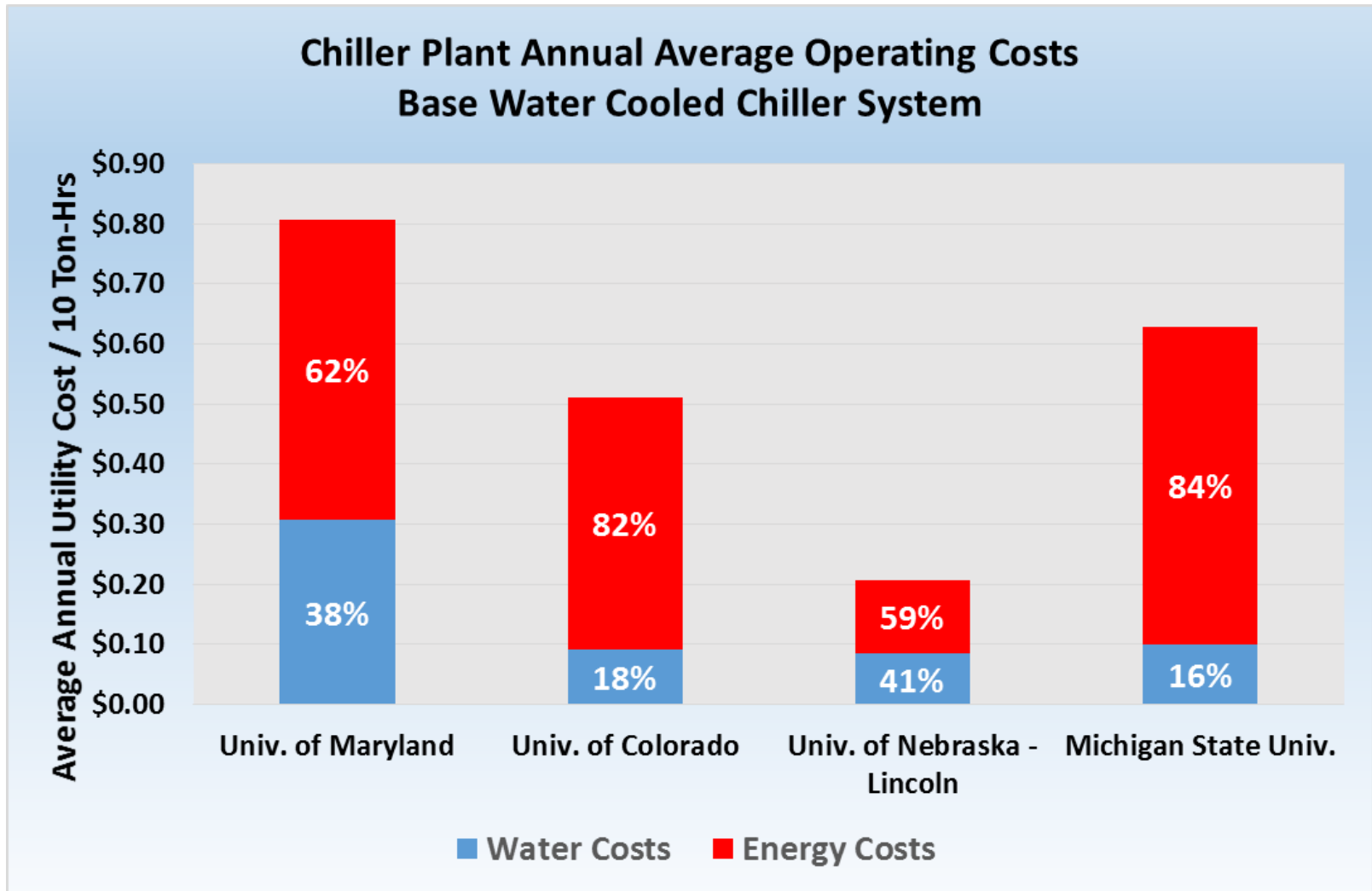
Location	Annual Average DB (°F)	Annual Average WB (°F)	Annual Cooling Ton-Hrs*	Blended Electrical Energy Rate (\$/kWh)**	Fully Burdened Water Costs*** (\$/1000 gal of Make-up)	Cooling Tower CoC
UMCP	57.4	51.1	5,404,091	\$0.0809	\$18.11	4.5
U. of CO - Boulder	50.5	40.1	4,474,109	\$0.0790	\$ 5.76	8.0
U. Nebraska - Lincoln	52.2	46.4	5,210,070	\$0.0204	\$ 5.29	5.0
Michigan State Univ.	47.7	43.3	4,928,143	\$0.0921	\$ 5.98	3.3

\* Load profiles generated based on 1600 ton peak load, 200 ton minimum load

\*\* An additional demand charge of \$5.28/kW per month was applied to all systems that exceeded the peak monthly kW of the base water cooled system.

\*\*\* Includes water, wastewater, and chemical treatment costs

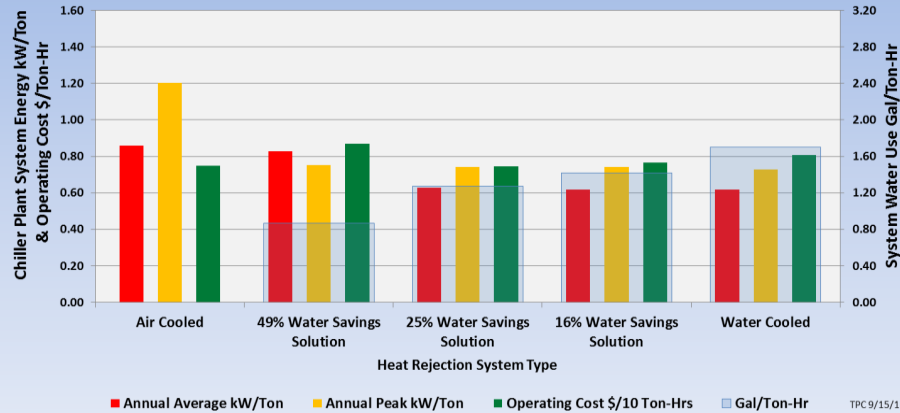
# Chiller Plant Average Annual Operating Cost Comparison



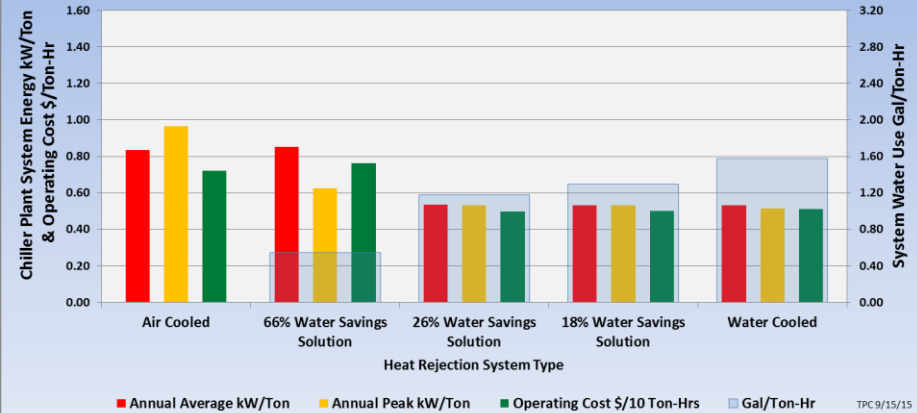
# Summary Across Four Universities



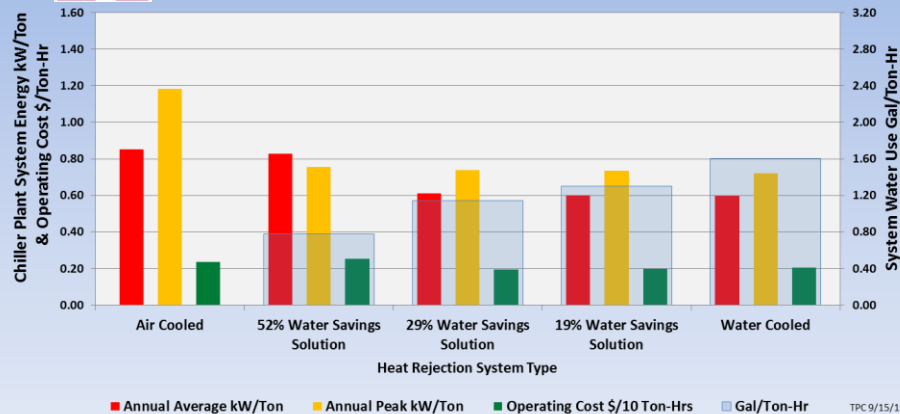
**Annual Peak and Average Energy Use, Operating Costs, and Annual Average Water Use For Various System Types**  
UMCP - Physical Sciences Building



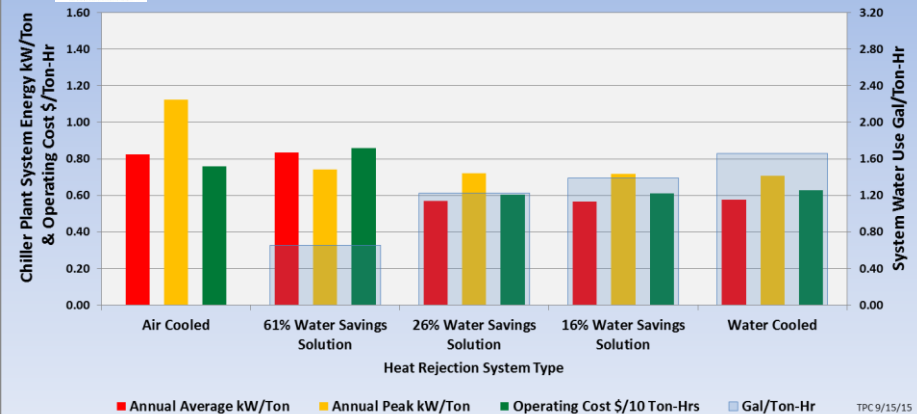
**Annual Peak and Average Energy Use, Operating Costs, and Annual Average Water Use For Various System Types**  
University of Colorado - Boulder



**Annual Peak and Average Energy Use, Operating Costs, and Annual Average Water Use For Various System Types**  
University of Nebraska - Lincoln



**Annual Peak and Average Energy Use, Operating Costs, and Annual Average Water Use For Various System Types**  
Michigan State University - East Lansing, MI





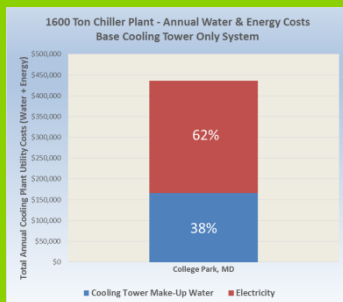
# Key Points From The Analysis:

---

- Across a wide range of climates and utility rates, hybrid heat rejection systems can save **both** water and annual utility costs.
- Water and utility operating cost savings are related to the number of dry cooling units installed.
- Using the same quantity of installed dry cooling equipment, a range of water savings can be achieved based on the operating strategy employed.
- As water related costs increase, the traditional operating cost advantage of water cooled systems compared to air cooled systems decreases.

# In Conclusion

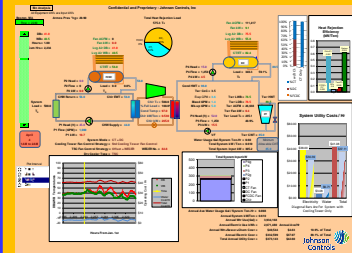
Water Costs Are Becoming An Increasing Larger Component of a Chiller Plant's Total Operating Cost



Drought and Water Availability Can Pose A Risk For Chiller Plant Operations



Analysis of Alternatives Requires a Thorough Annual System Evaluation



Hybrid Systems Offer a Cost Effective Way to Reduce Chiller Plant Water Use

