

District Energy and Water Source Heat Pumps

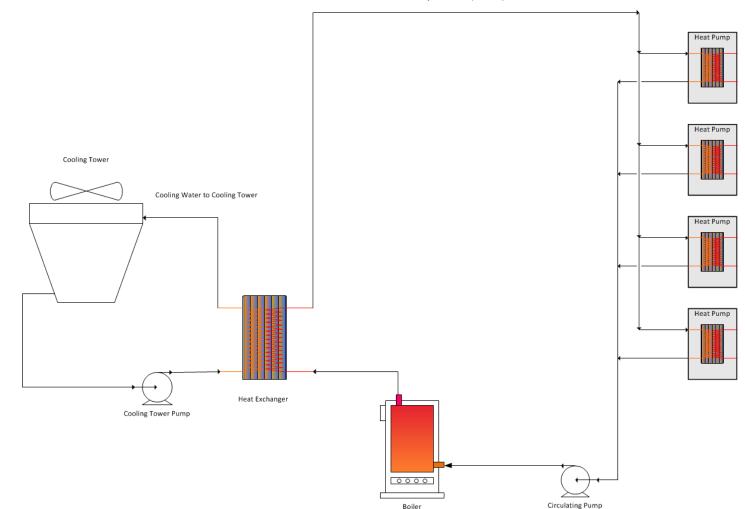
Issues and Opportunities

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Water Source Heat Pumps – The Challenge

- Developers like Water Source Heat Pumps (WSHP)
- District Cooling systems see them as a competitor to the traditional four-pipe system
- Using chilled water (CHW) to cool the condenser loop can have benefits for the developer, the building operator, and the district cooling supplier
- It is a difficult sale, so...
- A comprehensive approach must be taken

The Technology – Typical WSHP



Heat Rejection/Absorption Loop

Developer's Viewpoint

• Pros:

- Easy to install in new construction
- Relatively easy to install in retrofits/rehabs
- More efficient than air cooled heat pumps
- More efficient than split systems
- Better individual control of comfort
- Most of cost of heating/cooling is paid through the rental/condo unit's electric meter

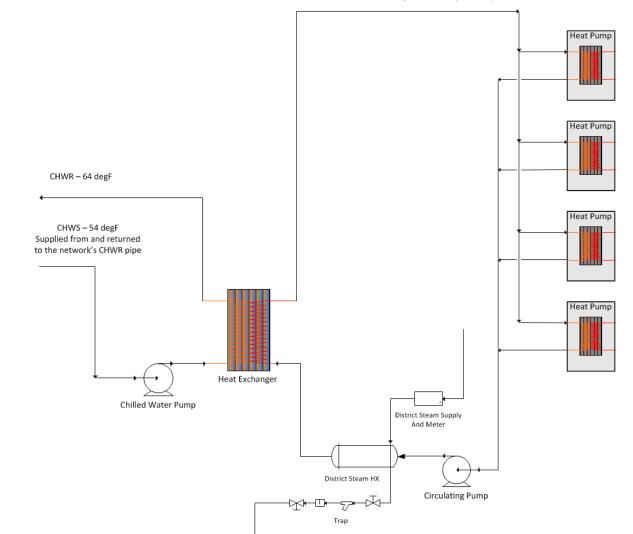
• Cons:

- Requires a boiler to heat the condenser loop
- Requires a cooling tower to cool the loop
- These use valuable space that could be used for additional revenue generation

District Energy Viewpoint

- We would much rather have the customer install a four pipe system
- But, four pipe systems are expensive to install, especially in retrofit/rehab situations
- So, what can we do?
 - Cool condenser loop with chilled water, to eliminate the cooling tower
 - Heat condenser loop with steam or hot water, eliminating the boiler
- What else can we do?
 - \circ Use chilled water from the return side of the system, to improve central plant ΔT
 - Use the ability to cool the condenser loop to lower than 85 °F to improve the efficiency of the WSHP units

WSHP with District Energy



Heat Rejection/Absorption Loop

Efficiency Benefits for Customer

| Assumptions | | | |
|--|----------|--------|--------|
| Performance Data from Trane EXH012 water | | | |
| # Units: | 1,200.00 | | |
| Loop Temperature | 55 | 86 | degF |
| gpm per unit | 2.90 | 2.90 | gpm |
| Total Cooling kbtuh | 12.98 | 11.90 | kbtuh |
| Total Heating kbtuh | 12.67 | 17.24 | kbtuh |
| Cooling compressor power - kW | 0.51 | 0.82 | kW |
| Heating compressor power - kW | 0.80 | 0.88 | kW |
| Cooling kW/Ton | 0.48 | 0.83 | kW/Ton |
| Heating kW/mbh | 0.06 | 0.05 | kW/mbh |
| DeltaP per floor (ft) | 4.50 | 4.10 | ft |
| Total DeltaP (ft) | 44.82 | 44.42 | ft |
| Circ Pump HP | 52.52 | 52.05 | HP |
| Rejected Heat - Cooling, kbtuh | 14.73 | 14.53 | kbtuh |
| Absorbed Heat - Heating, kbtuh | 9.95 | 14.25 | kbtuh |
| CT Pump HP @ 3 gpm/ton and 50 ft head | | 60.61 | HP |
| CT Fan kW | | 111.90 | kW |
| Total Demand: | 52.52 | 224.55 | kW |

Example – Western USA

| Customer kWh Saved: | 1,601,029 |
|--|---------------------|
| Veolia CHW Sales, Ton-Hrs | 3,019,536 |
| | |
| Total kWh Consumed | 1,677,270 |
| Circulating Pump kWh | 343,192 |
| Heat rejected to Veolia, kbtu | 36,234,432 |
| Heat added by Boiler, kbtu | - |
| Heat Absorbed from Loop, kbtu | 609,978 |
| Heat Rejected to Loop, kbtu | 36,844,410 |
| Total HP kWh | 1,334,598 |
| Heating kWh @ 55 Loop Temp | 48,921 |
| Cooling kWh @ 55 Loop Temp | 1,285,677 |
| Heating kWh @ 86 Loop Temp | 39,422 |
| Cooling kWh @ 86 Loop Temp | 2,242,686 |
| CC CHW | |
| Total kWh Consumed | 3,278,299 |
| Circulating Pump kWh | 340,129 |
| CT Pump kWh @ 25ft Head | 396,058 |
| Cooling Tower Fan kWh @ 0.1 kWh/ton-hr | 260,004 |
| Heat rejected to cooling tower, kbtu | 39,000,591 |
| Heat added by Boiler, kbtu | - |
| Heat Absorbed from Loop, kbtu | 642,015 |
| Heat Rejected to Loop, kbtu | 39,642,607 |
| Total HP kWh | 2,282,108 |
| Heating kWh @ 86 Loop Temp | 39,422 |
| Cooling kWh @ 86 Loop Temp | 2,242,686 |
| Self Perform | |
| | 770,720 |
| Customer Heating kbtu | 776,726 |
| Customer Ton-brs | |
| Customer Ton-hrs | Totals 2,705,592 |

Customer O&M Benefits

- These costs will be eliminated or substantially reduced:
 - Repair and replace Cooling Tower
 - Cooling Tower PM
 - Water treatment PM
 - Chemicals
 - Pump PM
 - Pump Replacement
 - Make-up to Cooling Tower
 - Sewer Charges

ΔT Improvement

| Plant Peak: | 10,000 | tons |
|-------------------------------|--------------|------|
| Design DeltaT: | 16 | degF |
| System Flow Rate: | 15,000 | gpm |
| WSHP Customer Tons: | 1,200 | tons |
| Customer Design DeltaT: | 10 | degF |
| Customer Flow Rate: | 2,880 | gpm |
| Actual Plant DeltaT: | 12 | degF |
| Actual System Flow Rate: | 20,000 | gpm |
| CHWST to Customer: | 54 | degF |
| CHWRT from Customer: | 64 | degF |
| Blended CHWRT after Customer: | 55.44 | degF |
| New system Flow Rate: | 17,857 | gpm |
| HP Savings at 150 psig: | 249 | HP |
| Savings at \$0.10/kWh | \$ 46,507 | |
| Savings/ton-hr: | \$ 0.02 | |

Economics

| Potential Revenue Additions | \$ 90,000 |
|-------------------------------------|---------------|
| Capital Plant Amortization | \$ 305,689 |
| O&M Savings | \$ 88,315 |
| Electric Savings @ \$0.10/kWh | \$ 160,103 |
| TOTAL | \$ 484,005 |
| | |
| CHW purchases @ \$0.15/ton-hr | \$ 452,930 |
| Less Credit for deltaT improvement: | \$ 46,507 |
| Net Cost of CHW to customer: | \$ 406,424 |
| | |
| NET savings to Customer: | \$ 77,581 |

Conclusions

- It's a difficult sale!
- Can be good for all parties
- Need to price the product creatively by sharing operational benefits with customer
- Definite life cycle benefits for the customer
- Definite operational and efficiency benefits for the DE Plant.

QUESTIONS?