



Advantages of Compound Cycle Cogeneration

27 February 2019



Agenda



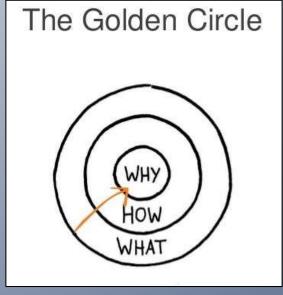
Start With Why

- System Comparison -Overview
- **Cogeneration Analysis Techniques**
- System Comparison Results
- Conclusions & Recommendations Question and Answers

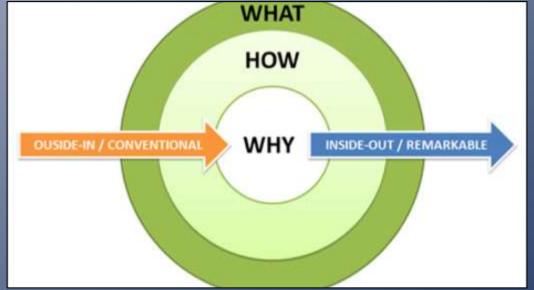


Start With Why

The Golden Circle:
1. Why = Our Cause, Our Beliefs
2. How = The Actions We Take
3. What = Daily Tasks

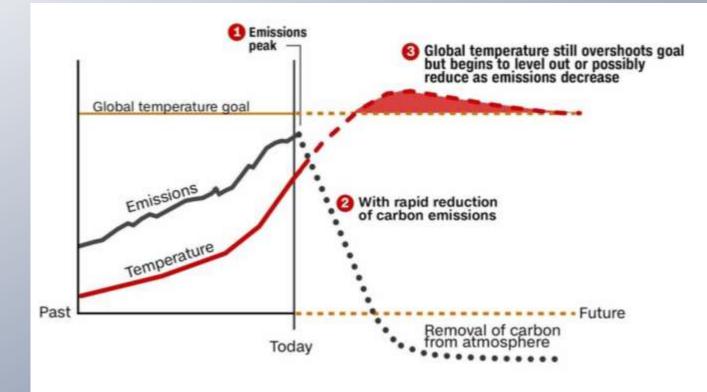


- Why = Deliver Energy More Efficiently
- "What Tools Do We Have?"
- "What Role Do We Take?"
- "What Information Do We Share?"





Business As Usual (BAU)

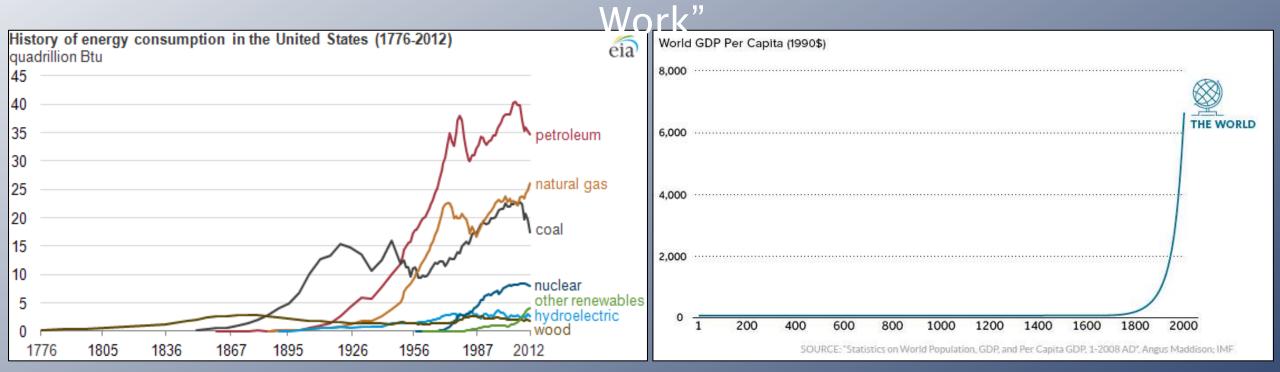






Why Energy Efficiency

"Energy Is Work Or The Capacity To Do





Impact Of Increased Efficiency

- Incandescent Bulb = 13 Lum/W
- Total Watts Required = 61 W
- Grid Efficiency = 95%
- Generated Watts = 64 W
- Generation Efficiency = 42.5%

• Total Watts Required = 151

- LED Fixture = 95 Lum/W
- Total Watts Required = 8 W
- Grid Efficiency = 95%
- Generated Watts = 8.4 W
- Generation Efficiency = 42.5%
- Total Watts Required = 19.8 W

One Lightbulb Retrofit Can Save 131 Watts



Start With Why

1. Second Law of Thermodynamics:

• "A Machine Whose Working Fluid Undergoes A Cycle Cannot Absorb Heat From A High Temperature Sink And Produce Shaft Work Without Rejecting Heat To A Lower Temperature Receiver."

2. More Effective Use of Rejected Heat:

 "The Obvious Way To Improve The Effectiveness Of Use Of The Input Energy, Then, Is To Put The Heat Rejected By The Cycle To Some Beneficial Use." - Bill Coad



Agenda

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Start With Why

System Comparison -Overview

Cogeneration Analysis Techniques

System Comparison - Results

Conclusions & Recommendations

Question and Answers



Compound Generation

- Burn Natural Gas To Generate Electricity (Gas Turbine)
- Utilize A Duct Burner and Heat Recovery Steam Generator
- Condense Steam To Generate Electricity (Steam Turbine Generator)
- Distribute Steam And Electricity To Campus

- Are All The Added Complexities Worth The Additional Step?
- One Field Trip Sparked The Following Analysis



System Descriptions

- 1. Conventional System
 - Purchase Electricity And Natural Gas
- 2. System "X"
 - Purchase Natural Gas To Generate Electricity
- 3. System "Y"
 - Compound Cycle With Condensing Steam (Mizzou Setup)
 - Solar Titan 130-20500 Gas Turbine (13.5 MW @ 13.8kV)
- 4. System "Z"
 - Compound Cycle Without Condensing Steam By Cooling Towers
 - Matching Campus Heating Needs And Heating Production Exactly



System Descriptions - Assumptions

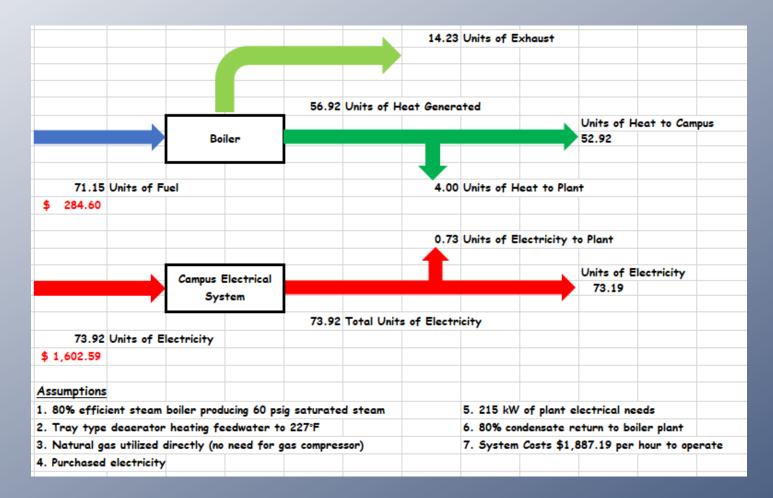
System	Electrical Output (kW)	Annual Electrical Output (kWh)	Annual Natural Gas Use (MMBtu)	Recovered Heat (MMBtu)
System "X"	13,477	112,155,594	1,404,060	440,580
System "Y"	21,446	178,470,277	2,014,204	440,580
System "Z"	18,429	153,365,527	2,014,060	799,375

Plant Assumptions and		
Natural Gas Cost =	\$4.00	per MMBtu
Purchased Electricity Cost =	\$0.074	per kWh
Steam Utilization =	100%	
Electrical Utilization =	100%	
Plant Availability =	95%	

Annual Plant Utility Production		
Annual Steam Production =	957,717,000	lbs
Annual Electrical Production =	216,266,000	kWh
Cooling Production =	57,071,426	ton-hours
Annual Biomass Use =	120,000	tons
Assumed Biomass Combustion Efficiency =	72%	
Assumed Gas Boiler Combustion Efficiency =	80%	
Assumed Biomass Heat content =	5,160	Btu/Ib
Annual Biomass Heat Production =	891,648	MMBtu
Annual Biomass Steam Production =	814,513,565	lbs



Conventional System

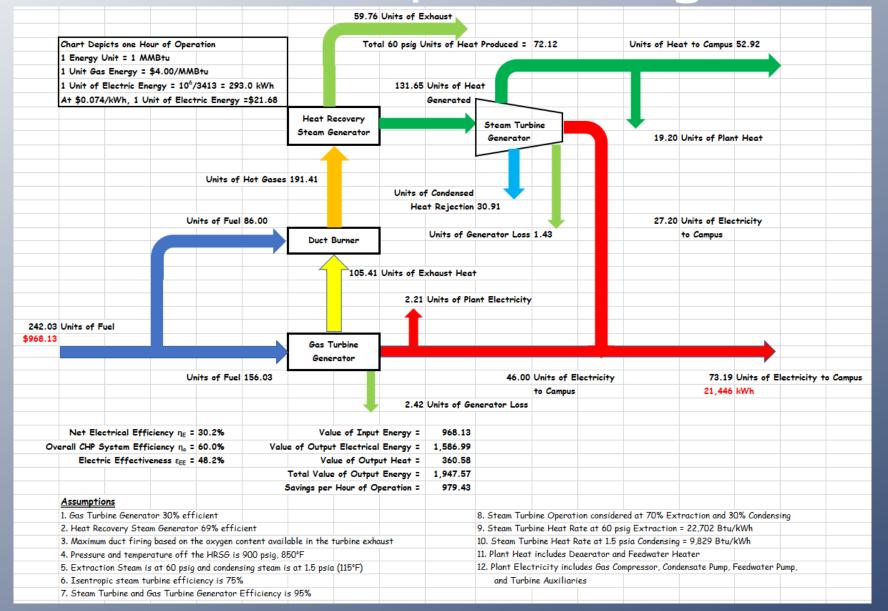




System "X" – Standard Cogeneration

Chart Depicts one Hour of Ope	eration	53.57	Units of Exhaust							
1 Energy Unit = 1 MMBtu										
1 Unit Gas Energy = \$4.00/M/	MBtu									
1 Unit of Electric Energy = 10	⁶ /3413 = 293.0 kWh									
At \$0.074/kWh, 1 Unit of Ele										
	, , ,, ,, ,, ,, ,, ,, ,, ,, ,, ,, ,, ,,		65.04 Units of H	laat Produced						
			00.04 01113 01 1	lear moduced						
		Heat Recovery								
		Steam Generator								
					50.00	11-14-14-14				
					52.92	Units of H	eat to Ca	npus		
	Units of Hot Gases	118.61	12.1	Units of Hea	t to Plant					
	Units of Fuel 12.68									
		Duct Burner								
		Duct burner								
		105.93	Units of Exhaust He	at						
			1.65 Units of P	lant Electricit	y					
Units of Fuel										
168.72								_		
		Gas Turbine								
\$ 674.87		Generator								
• • • • • • • • • • • • • • • • • • • •	Units of Fuel 156.00			46.00	Units of E	lectricity				
				\$ 997.21			LML			
			2.42 Units of G	-		13,470	KWA			
		•	2.42 UNITS OF G	enerator Loss						
Assumptions										
1. Gas Turbine Generator 30% et				8. Steam Tur						idensing
2. Heat Recovery Steam Generat				9. Steam Turbine Heat Rate at 60 psig Extraction = 22,702 Btu/kWh 10. Steam Turbine Heat Rate at 1.5 psia Condensing = 9,829 Btu/kWh						
3. Maximum duct firing based on			st						Btu∕kWh	
Pressure and temperature off			11. Plant Heat							
5. Extraction Steam is at 60 ps		at 1.5 psia (115°F)		12. Plant Elec			mpressor, (Condensate Pi	ump, Feedwa	ter Pump
6. Isentropic steam turbine effi	· ·			and Turbi	ne Auxiliari	ies				
7. Steam Turbine and Gas Turbin	ne Generator Efficiency is	95%								
Cost of Fuel	= \$ 674.87									
Value of Electricity Produced	= \$ 997.21									
Value of Heat Produced	= \$ 264.60									
Savings per Hour of Operation										-
						1		1		1

System "Y" – Compound Cogeneration



System "Z" – Compound Cogeneration

			59.76 Units of	f Exhaust						
Chart Depic	ts one Hour of Operation		Total 60 psi	g Units of Hea	t Produced =	113.86	Units of	Heat to Campus	96.06	
1 Energy Ur	nit = 1 MMBtu									
	Energy = \$4.00/MMBtu									
	lectric Energy = 10 ⁶ /3413 =	293 0 LWL	121	65 Units of He						
	••		151.							
AT \$0.074/	kWh, 1 Unit of Electric Ener	rgy =\$21.68		Generated						
			Heat Recovery		Steam Tur					
			Steam Generator					80 Units of Plan		
					Generator		17.	80 Units of Plan	t Heat	
	Units	of Hot Gases 1	91.41							
	Units of F	Fuel 86.00					16	90 Units of Elec	tricity	
				Unite of G	enerator Loss	0.89	10.	to Campus		
			Duct Burner			0.02		re campas		
			105.41 Units of	f Exhaust Hea	•					
			2.	21 Units of Pla	ant Electricity	/				
242.03 Units of Fue	-1									
968.13	51									
00.13			Gas Turbine							
			Generator							
	Unite of F	uel 156.03			46.00	Units of Electi	alalah (62.00	Units of Elec	tricity to Camp
	Units of I	uei 130.03			40.00		псту			Tricity to camp
						to Campus		18,429	ĸwn	
			2.	42 Units of Ge	nerator Loss					
Net Elect	trical Efficiency η_E = 26.0%		Value of Input Energy	= 968.13						
Overall CHP Sy	stem Efficiency no = 73.0%	Value	of Output Electrical Energy	= 1,363.75						
Electric	: Effectiveness ε _{EE} = 63.1%		Value of Output Heat	= 569.30						
		т	otal Value of Output Energy	= 1,933.06						
		Se	vings per Hour of Operation	= 964.92						
Assumption	IS									
	≌ ne Generator 30% efficient				8 Steam Tur	hine Operation 1	00% Extraction ar	d 0% Condensing		
	overy Steam Generator 69% et	fficient					t 60 psig Extracti			
	duct firing based on the oxyge		le in the tunhine exhaust				at 1.5 psia Condens			
3 Maximum							at 1.5 psia condens ator and Feedwater	-		
	4. Pressure and temperature off the HRSG is 900 psig, 850°F					menudes Deder				
4. Pressure o					12 Dlan+ El-	والبيا ومتريطته تمطر	Gas Company C	andonaata D	Condunator D	-
4. Pressure o 5. Extractio	n Steam is at 60 psig and con	densing steam is					Gas Compressor, C	ondensate Pump,	Feedwater Pum	p,
4. Pressure o 5. Extractio 6. Isentropio		densing steam is 60%	at 1.5 psia (115°F)			tricity includes ine Auxiliaries	Gas Compressor, C	ondensate Pump,	Feedwater Pum	p,

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- **Cogeneration Analysis Techniques**
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Typical Cogeneration Analysis Techniques

. Single Equation For Cogeneration

- Developed By Bill Coad
- Approach To Producing Chilled Water
- 2. Life Cycle Cost Analysis
 - Published in ASHRAE by Michael Schwarz
- 3. "Best Case" Simple Payback
 - The Jerry Williams Approach
 - First Pass Estimate (With Great Detail)
- 4. Monte Carlo Simulation
 - Since No System Is Ever Static



Single Equation For Cogeneration $K_G = F(10^{-4}) \left[R - \left(\frac{1}{\lambda_B}\right) (H_r U_H) \right] - \left(\frac{1}{A_R}\right) (E_R U_C H_R K_P) + M + I + X$

- K_G = cost to generate electricity, cents per kWh
- R = prime mover fuel rate, Btu per kWh
- F = cost of fuel, \$ per million Btu
- H_r = salvage heat available, Btu per kWh
- λ_B = boiler efficiency in producing heat from fuel
- U_H = utilization ratio for recovered heat
- M = cost of maintenance, cents per kWh
- I = amortized investment cost, cents per kWh

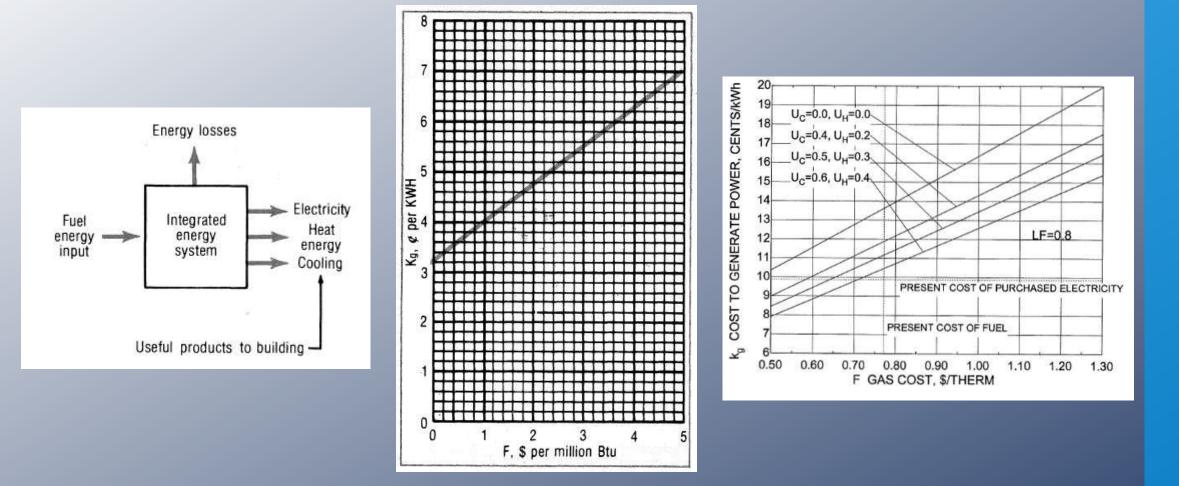
X = any other fixed cost, cents per kWh $A_R =$ fluid heat rate for absorption cooling, Btu per ton-hr

E_R = energy requirement for compression refrigeration, Kw per ton

- K_P = cost of purchased electricity, cents per kWh generated
- U_C = utilization ratio for heat recovered for cooling



Single Equation For Cogeneration





	PURCHASED	CHP	
	ELECTRICITY (Existing Condition)	(On-Site 2500 kW CHP)	NOTES
INIT DATA			
Installed Power Generation Capacity (kW)	N/A	2500	CHP Size
Average Annual Energy Available (kWh)	21,900,000	21,900,000	
CapEx			
Total Installation Costs	\$0	\$3,750,000	Assumes \$1500/kW Installed CHP Plant Capacity and That System is Being Installed as a Retrofit. Therefore, No Capital cost Offset is Assumed for Displaced Boilers or Other Equipment
DpEx .			
Electric Utility Rate (\$/kWh), Consumption	\$0.11	\$0.11	
Electric Utility Rate (\$/kW), Demand	\$2.35	\$2.35	Monthly Peak Demand Rate
Electric "Sell-Back" Rate for Unused Power (\$/kWh)	-	\$0.06	
Natural Gas Utility Rate (\$/therm)	\$0.91	\$0.91	
CHP System Availability (% of year)	-	90%	
uilding Power Utilization (% Annual Generation)	-	75%	
Annual Electric Consumption/Generation (kWh)	14,782,500	-19,710,000	Purchased Electricity Cost Includes Monthly
Annual Electric Cost (\$)	\$1,696,575	-\$295,650	2,500 kW Additional Demand
Annual Gas Consumption (therms)		2,069,333	Assumes 32.5% Power Generation Efficiency
Annual Gas Cost (\$)	2	\$1,883,093	
Annual Waste Heat Available (Mbtu)	-	96,076,170	Thermal Credit. Assumes CHP Power-to-Heat Ratio of 0.7
Waste Heat Thermal Utilization (%)	75%	-	Measure of Available CHP Thermal Energy That is Used
Annual Waste Heat Equivalent Gas Cost (\$)	\$1,008,800	-	Displaced Thermal Fuel Cost. Assumes 65% Tota Efficiency for Steam Boiler Plant Operation in Base Case for Energy Comparison
Annual Maintenance Cost (\$)	\$0	\$295,650	\$0.015/kWh Maintenance Cost
Total Year 1 OpEx	-\$2,705,375	-\$1,883,093	
Incentive(s)	N/A	\$375,000	Federal Renewable Energy Investment Tax Credit of 10% of Construction Cost Included in Year 1 (No Maximum Credit)
Unit Replacement Cost	N/A	N/A	

Energy Solutions

Powered for good.

Life Cycle Cost Analysis Michael Schwarz

Discount Rate	15%	15%	The Interest Rate Used for Discounting (Rate That Reflects an Investor's Opportunity Cost of Money Over Time and Minimum Acceptable Rate of Return)
Operating Cost Inflation Rate	5%	5%	
Initial Cost of Investment	\$0	-\$3,375,000	
2	-\$2,840,644	-\$1,977,248	OpEx - Year 2, Adjusted for Inflation
3	-\$2,982,676	-\$2,076,110	OpEx - Year 3, Adjusted for Inflation
YEAR	S 4 THROUGH 19 H	IDDEN FOR CLARIT	Ŷ
20	-\$6,836,347	-\$4,758,482	OpEx - Year 20, Adjusted for Inflation
Simple Payback (Years), No Incentives	-	4.6	Total Installation Costs/Difference In Total Year 1 OpEx
Simple Payback (Years), with Incentives Included	-	4.1	(Total Installation Costs-Incentive)/Difference In Total Year 1 OpEx
5-Year NPV (With Incentives)	-\$11,370,200	-\$11,289,298	Year 1-5
10-Year NPV (With Incentives)	-\$17,391,990	-\$16,311,217	Year 6-10
15-Year NPV (With Incentives)	-\$23,163,091	-\$19,497,813	Year 11-15
20-Year NPV (With Incentives)	-\$26,068,050	-\$21 519 828	Year 15-20

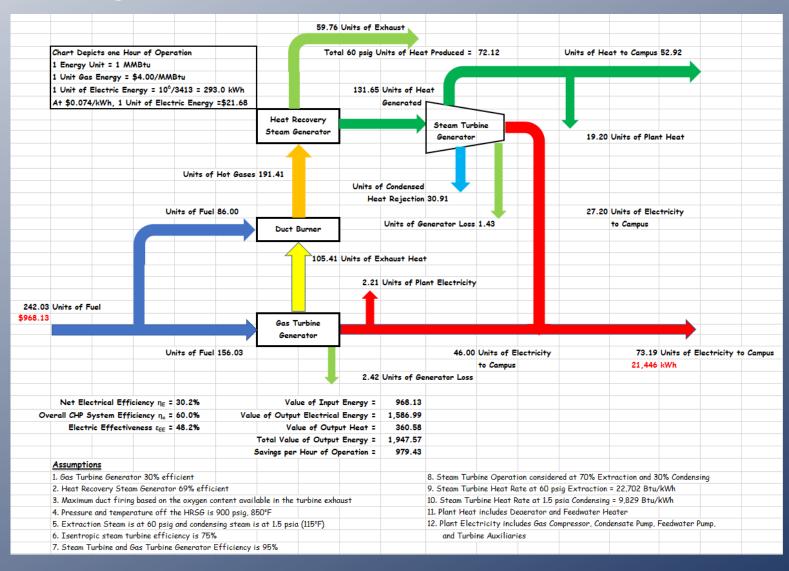
- Published in August 2017 ASHRAE Journal
- Comparison To Business-As-Usual (BAU)
- 20-Year Timeline

Life Cycle Cost Analysis Michael Schwarz (cont.)



Best Case Simple Payback

- Perfected By Jerry Williams
- Detailed Approach
- "Must Draw It To Understand It"
- Savings Over First Cost
- First Pass Analysis
- Typically Requires Further Analysis Before Final Recommendation (But Not Much)



Best Case Simple Payback

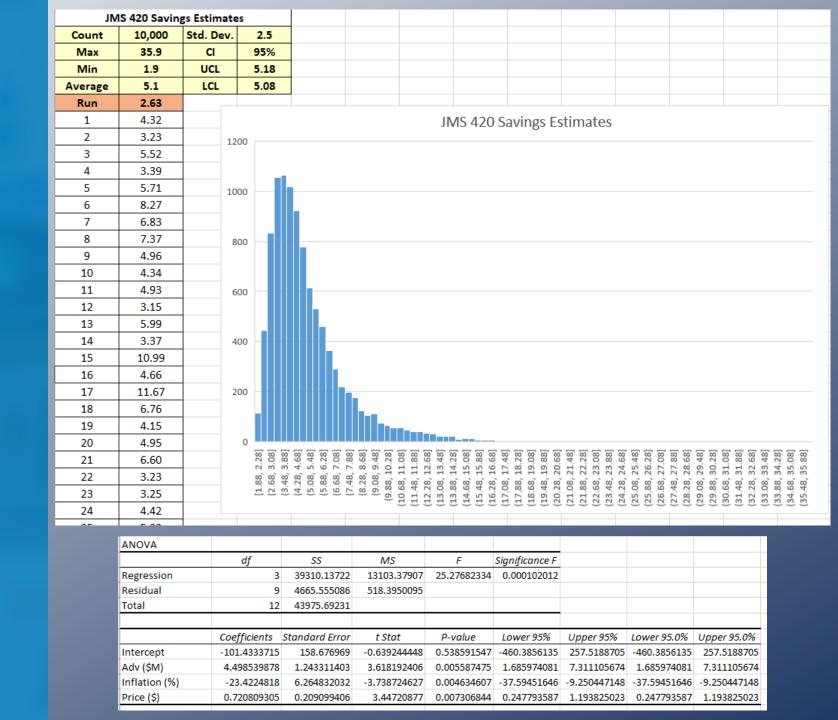
	-	Combin	ed Cycle f	Powerplar	nt Perfor	rmance C	haracterist	ics - Opti	ion X -	Maximum	Duct Fir	ring			
Media	Pressure (psig)	Pressure (psia)	Temperature (°F)	Quality (%)	Enthalpy (Btu/lb)	Entropy (Btu/lb-°R)	Specific Volume (ft ³ /lb)	Specific Heat (Btu/lb-°R)	Heat Rate (Btu/kWh)	Steam Rate (lb/kWh)	Flow (scfh)	Flow (lb/hr)	HHV Flow (MMBtu/hr)	Voltage (kV)	Power (kW)
Total Natural Gas	50										231,833		242.0		
Natural Gas to Turbine	343										149,458		156.0		
Ambient Air		14.7	55												
Flue Gas Off Gas Turbine			944					0.280				384,450			
Superheated Steam	900		850	Steam	1,422.7	1.602	0.798								
Superheated Steam	60		375	Steam	1,218.7	1.673	6.455		22,702	18.48					
Steam Off Turbine		1.5	115	93.0	1,026.0	1.944	228.435		9,829	9.43					
Saturated Condensate		1.5	115	Liquid	83.0	0.156	0.016								
Subcooled Condensate				Liquid	83.0	0.156	0.016								
Subcooled Condensate		14.7	180	Liquid	148.0	0.263	0.017								
Subcooled Condensate		14.7	159	Liquid	124.0	0.225	0.016								
Saturated Condensate	5		227	Liquid	195.4	0.335	0.017								
Subcooled Condensate	900		227	Liquid	197.4	0.333	0.017								
Subcooled Condensate	900		300	Liquid	271.3	0.436	0.017								
City Water		14.7	60	Liquid	28.1	0.056	0.016								
Superheated Steam	5		380	Steam	1,228.3	1.784	16.653								
Generated Electricity														13.8	7,969
Generated Electricity														13.8	13,477
Generated Electricity														13.8	21,446
Flue Gas Off HRSG			422									384,450			
Natural Gas to Duct Burner											82,375		86.0		
Natural Gas to Compressor													156.0		
Flue Gas Off Duct Burner			1,645												
Steam Turbine Efficiency =	0.78		Gas Turbine H	leat Rate (HHV	- Btu/kWh) =	11,578	Ste	am Turbine Genero	ation from Ext	raction (kW) =	4,331	HR	SG Efficiency =	68.8%	
· ·		Go				156.0							,		
~						46.0									
· · ·						2.4						Addit	ional Capacity =	59.1%	
· · · ·						86.0							. ,		
						191.4					29.5%				
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	10.8					131.7								1	
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	6.800				· · ·	0.70									
N 1 1							Remaining Tur			N					
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	75.37				21										
	Total Natural Gas Natural Gas to Turbine Ambient Air Flue Gas Off Gas Turbine Superheated Steam Superheated Steam Steam Off Turbine Saturated Condensate Subcooled Condensate Subconde Subcooled Conde	Media Pressure (psig) Total Natural Gas 50 Natural Gas to Turbine 343 Ambient Air 343 Flue Gas Off Gas Turbine 343 Superheated Steam 900 Superheated Steam 900 Superheated Steam 900 Superheated Steam 60 Steam Off Turbine 343 Subcooled Condensate 5 Subcooled Condensate 5 Subcooled Condensate 900 City Water 5 Subcooled Steam 5 Generated Electricity 6 Generated Electricity 10 Generated Electricity 10 Natural Gas to Duct Burner 10.36 Natural Gas to Compressor 10.37 Flue Gas Off Duct Burner 0.96 steam Turbine Efficiency	MediaPressure (psig)Pressure (psig)Total Natural Gas50Natural Gas to Turbine343Ambient Air14.7Flue Gas Off Gas Turbine343Superheated Steam900Superheated Steam60Steam Off Turbine1.5Saturated Condensate1.5Subcooled Condensate14.7Subcooled Condensate14.7Subcooled Condensate900Subcooled Condensate900Subcooled Condensate14.7Subcooled Condensate900Subcooled Subcooled Steam5Generated Electricity90Flue Gas Off HRS690Natural Gas to Duct Burner90Steam Turbine Efficiency =0.78Generator Efficiency =0.75	MediaPressure (psig)Temperature (psid)Total Natural Gas50Natural Gas to Turbine343Ambient Air14.7Flue Gas Off Gas Turbine900Superheated Steam900Superheated Steam603turated Condensate1.5Staurated Condensate14.7Subcooled Condensate14.7Subcooled Condensate14.7Subcooled Condensate227Subcooled Condensate5Subcooled Condensate3800City Water14.7Subcooled Condensate5Subcooled Condensate3800Generated Electricity6Generated Electricity14.7Generated Electricity14.7Generated Electricity14.7Flue Gas Off HRSG422Natural Gas to Duct Burner1.645Steam Turbine Efficiency =0.78Generate Efficiency =0.75Equivalent Efficiency =0.75Generator Efficiency =0.75Generator Efficiency =0.75Generator Efficiency =0.75Generator Efficiency =0.75Generator Stengues =80%Duct124.0Total Natural Gas to Use (lb/hr) =18.80Final Exhaustor Heat Required (MMBtu/hr) =10.8Exhaust10.8Exhaust6or Heat Required (MMBtu/hr) =38.8Turbine exit steam energy flow1.5 pisia trtrate *(h_b - 277.0) (MMBtu/hr)75.37	MediaPressure (prig)Pressure (prig)Temperature (°F)Quality (%)Total Natural Gas50Natural Gas to Turbine343Ambient Air14.755Flue Gas Off Gas Turbine900850SteamSuperheated Steam900850SteamSuperheated Steam60375SteamSteam Off Turbine1.5115LiquidSubcooled Condensate1.5115LiquidSubcooled Condensate14.7180LiquidSubcooled Condensate14.7180LiquidSubcooled Condensate900227LiquidSubcooled Condensate900300LiquidSubcooled Condensate900300LiquidSubcooled Condensate900300LiquidSubcooled Condensate900300LiquidSubcooled Condensate900300LiquidSuperheated Steam5380SteamGenerated ElectricityGenerated ElectricityFlue Gas Off Duct BurnerNatural Gas to CompressorFlue Gas Off Duct BurnerNatural Gas to CompressorFlue Gas Off Duct BurnerNatural Gas to CompressorSteam Turbine Efficiency =0.75Equivalent Heat Rate (HHV -nall Steam Turbine Efficiency = <td>Media Pressure (prig) Temperature (prig) Coulity (%) Enthalpy (Btu/lb) Total Natural Gas to Turbine 50 </td> <td>Media Pressure (psig) Feasure (psig) Temperature (r) Quality (%) Entholpy (Btv/lb-) Entropy (Btv/lb-) Total Natural Gas to Turbine 343 </td> <td>Media Pressure (pig) Pressure (pig) Temperature (pig) Quality (%) Enthalpy (Bux/b) Entropy (Bux/b-R) Specific Volume (bux/b-R) Total Natural Gas 343 </td> <td>Madia Pressure (prip) (prip) Pressure (prip) Temperature (prip) Quality (%) Entropy (Bru/b) Entropy (Bru/b-R) Specific Velume (pri/b) Specific Velume (pri/b)</td> <td>Madia Pressure (prig) Freesure (prig) Temperature (prig) Construction (prig) Entropy (Bru/lb) Specific Values (Bru/lb-/R) Specific Values (Bru/lb-/R)</td> <td>Madia Pressure (pip) Temperature (pip) Temperature (pip) Quality (%) Enthalpy (Btr/B-16) Specific Value (H¹/B) Specific Value (H¹/B) Specific Value (H¹/B) Heat Rate (Br/M-N) Heat Rate (Br/M-N) Total Natural Gas 50 (Br/M-N) (Br/M</td> <td>Media Pressure (part) Temperature (r) Coultry (N) Entainy (PL/Rs) Entropy (PL/Rs) Specific Volum (PL/Rs) Specific Volum (PL/Rs) Specific Volum (PL/Rs) Heat Rate (PL/Rs) Heat</td> <td>Media (pri) (pri) (pri) (priv) (priv)</td> <td>Media Pressure (pick) Pressure (pick) Temperature (pick) Entropy (PU/Dev/Dev/Dev/Dev/Dev/Dev/Dev/Dev/Dev/Dev</td> <td>Medic Presson Construction Description Construction Standhiston Standhiston Presson Construction Presson <t< td=""></t<></td>	Media Pressure (prig) Temperature (prig) Coulity (%) Enthalpy (Btu/lb) Total Natural Gas to Turbine 50	Media Pressure (psig) Feasure (psig) Temperature (r) Quality (%) Entholpy (Btv/lb-) Entropy (Btv/lb-) Total Natural Gas to Turbine 343	Media Pressure (pig) Pressure (pig) Temperature (pig) Quality (%) Enthalpy (Bux/b) Entropy (Bux/b-R) Specific Volume (bux/b-R) Total Natural Gas 343	Madia Pressure (prip) (prip) Pressure (prip) Temperature (prip) Quality (%) Entropy (Bru/b) Entropy (Bru/b-R) Specific Velume (pri/b) Specific Velume (pri/b)	Madia Pressure (prig) Freesure (prig) Temperature (prig) Construction (prig) Entropy (Bru/lb) Specific Values (Bru/lb-/R) Specific Values (Bru/lb-/R)	Madia Pressure (pip) Temperature (pip) Temperature (pip) Quality (%) Enthalpy (Btr/B-16) Specific Value (H ¹ /B) Specific Value (H ¹ /B) Specific Value (H ¹ /B) Heat Rate (Br/M-N) Heat Rate (Br/M-N) Total Natural Gas 50 (Br/M-N) (Br/M	Media Pressure (part) Temperature (r) Coultry (N) Entainy (PL/Rs) Entropy (PL/Rs) Specific Volum (PL/Rs) Specific Volum (PL/Rs) Specific Volum (PL/Rs) Heat Rate (PL/Rs) Heat	Media (pri) (pri) (pri) (priv) (priv)	Media Pressure (pick) Pressure (pick) Temperature (pick) Entropy (PU/Dev/Dev/Dev/Dev/Dev/Dev/Dev/Dev/Dev/Dev	Medic Presson Construction Description Construction Standhiston Standhiston Presson Construction Presson Presson <t< td=""></t<>

Monte Carlo Simulation (Because No System Is Static)

- Technique Used To Understand The Impact Of Risk And Uncertainty
- Generate Draws From A Probability
 Distribution
- Require:
 - Input Variables (With Ranges)
 - Probability Of Different Outcomes
 - Repeated Random Sampling
- Deliver:
 - Mean And Standard Deviation
 - Confidence Interval
 - Regression Capabilities



Monte Carlo Simulation (Because No System Is Static)



Agenda



Start With Why

- System Comparison -Overview
- **Cogeneration** Analysis Techniques
- System Comparison Results
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Typical Cogeneration Analysis Techniques 3. "Best Case" Simple Payback • The Jerry Williams Approach First Pass Estimate (With Great Detail)

4. Monte Carlo Simulation

• Since No System Is Ever Static



Pro Forma Results

Net Of Chilled Water Production

System	Value Of Electricity Generated	Value Of Recovered Steam	Net Annual Savings	Total Plant Cost	Payback Period	Cost / kWh Produced
System "X"	\$8,299,514	\$2,202,901	\$3,652,461	\$30,323,250	8.3	\$0.0414
System "Y"	\$13,206,801	\$2,201,834	\$5,388,642	\$39,487,139	7.3	\$0.0438
System "Z"	\$11,349,049	\$3,996,876	\$5,602,085	\$33,740,079	6.0	\$0.0375

With Absorption Refrigeration Cost And Savings

System	Absorption Chiller Capacity Available (Tons)	Value Of Ton- Hours Produced	Value of Additional Electricity	Net Annual Savings	Total Plant Cost	Paybac k Period	Cost / kWh Produced
System "X"	6,089	\$2,493,526	\$434,961	\$3,418,401	\$33,367,658	9.8	\$0.0435
System "Y"	6,086	\$2,495,228	\$435,258	\$5,104,002	\$42,530,072	8.3	\$0.0454
System "Z"	11,047	\$4,524,177	\$789,180	\$5,217,514	\$39,263,760	7.5	\$0.0400



Agenda



Start With Why

- System Comparison -Overview
- Cogeneration Analysis Techniques
- System Comparison Results

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Conclusions & Recommendations

Question and Answers



Conclusions

- 1. Never Forget The Opportunity Cost Of Electric Refrigeration
 - The Value Of The Recovered Steam For Absorption Refrigeration Is Of Less Value Than The Steam Recovered For Heating
- 2. Steam Turbine Addition Reduces Payback By 10%
- 3. What If Plant Operating Pressure Was 600 PSIG?
- 4. What If The Steam Turbine Exhausted At 60 PSIG?
- 5. What Are The Affects Of Standby Service Riders?
 - How Is Generation Calculated?



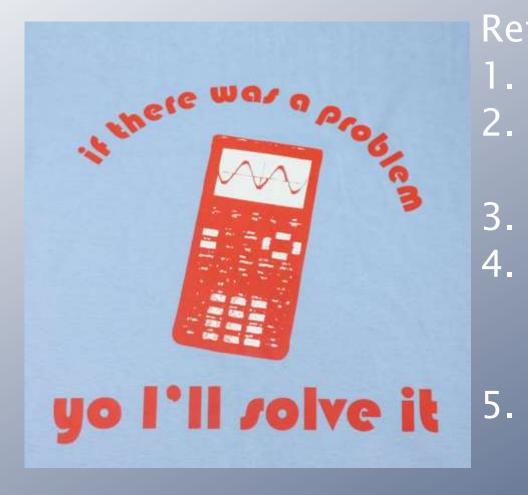
Agenda



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Ouestion & Answers



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